

II - Models

Notation



SMME/ESO
Skipsmodelltanken

Technical and Research Bulletin No. 1-5

Nomenclature for Treating the Motion of a Submerged Body Through a Fluid

Report of the American Towing Tank Conference
prepared by the Hydromechanics Subcommittee
of the Technical and Research Committee
of The Society of Naval Architects and
Marine Engineers

Published in April, 1950, by
The Society of Naval Architects and Marine Engineers
29 West 39th Street, New York 18, N. Y.

NOMENCLATURE FOR MOTION OF SUBMERGED BODY THROUGH A FLUID 15

The body is said to be in the direction of motion. The body is said to be in a steady state when the position of the body is constant relative to the fluid. The body is said to be in a steady state when the position of the body is constant relative to the fluid. The body is said to be in a steady state when the position of the body is constant relative to the fluid.

Dynamic stability or **Course**. A body is said to be dynamically stable or course if, after it is displaced from its equilibrium position, it returns to that position after a finite time. A body is said to be dynamically unstable if, after it is displaced from its equilibrium position, it does not return to that position after a finite time. A body is said to be dynamically neutral if, after it is displaced from its equilibrium position, it remains in a new position.

Symbol	Typical non-dimensional formula	Definition of symbol
a	$a = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
b	$b = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
c	$c = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
d	$d = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
e	$e = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
f	$f = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
g	$g = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
h	$h = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
i	$i = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
j	$j = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
k	$k = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
l	$l = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
m	$m = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
n	$n = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
o	$o = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
p	$p = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
q	$q = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
r	$r = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
s	$s = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
t	$t = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
u	$u = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
v	$v = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
w	$w = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
x	$x = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
y	$y = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
z	$z = \Delta h/g$	Derivative with respect to time of a characteristic length of a body

Symbol	Typical non-dimensional formula	Definition of symbol
g	$g = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
h	$h = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
i	$i = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
j	$j = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
k	$k = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
l	$l = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
m	$m = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
n	$n = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
o	$o = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
p	$p = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
q	$q = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
r	$r = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
s	$s = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
t	$t = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
u	$u = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
v	$v = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
w	$w = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
x	$x = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
y	$y = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
z	$z = \Delta h/g$	Derivative with respect to time of a characteristic length of a body

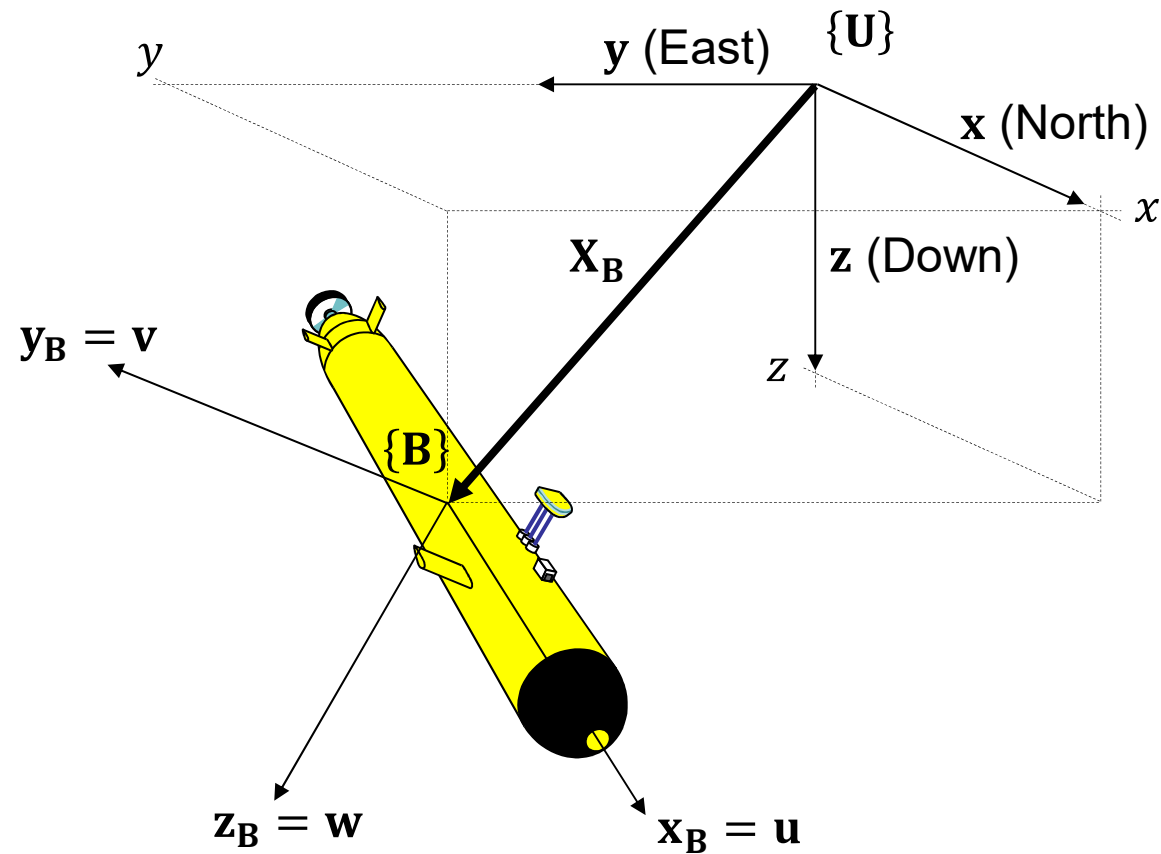
Symbol	Typical non-dimensional formula	Definition of symbol
g	$g = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
h	$h = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
i	$i = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
j	$j = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
k	$k = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
l	$l = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
m	$m = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
n	$n = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
o	$o = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
p	$p = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
q	$q = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
r	$r = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
s	$s = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
t	$t = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
u	$u = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
v	$v = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
w	$w = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
x	$x = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
y	$y = \Delta h/g$	Derivative with respect to time of a characteristic length of a body
z	$z = \Delta h/g$	Derivative with respect to time of a characteristic length of a body

II – Models

a) Representation Formalism

Representation Formalism

- Position



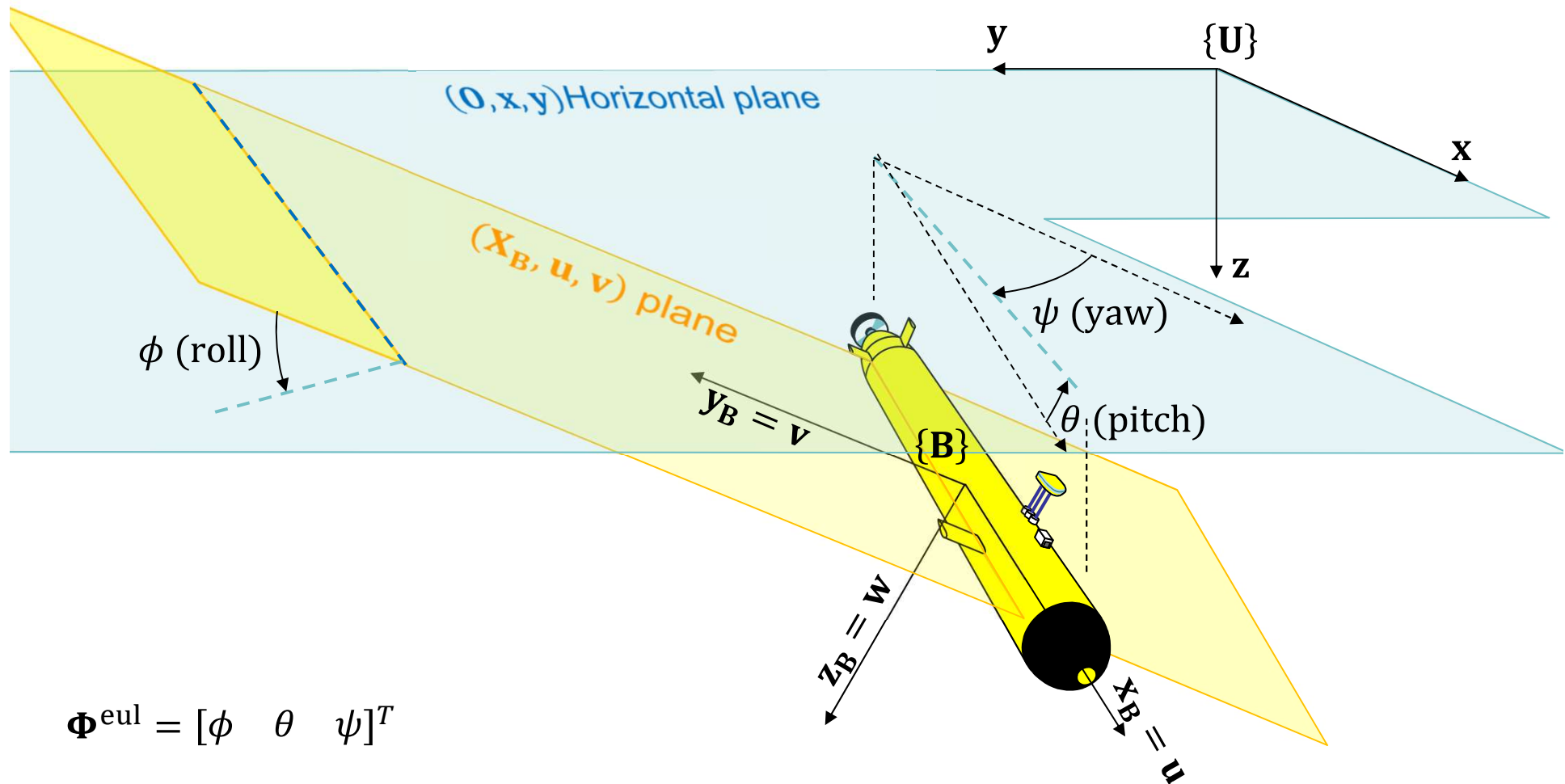
$$\{U\}: (\mathbf{0}, \mathbf{x}, \mathbf{y}, \mathbf{z})$$

$$\{B\}: (\mathbf{X}_B, \mathbf{u}, \mathbf{v}, \mathbf{w})$$

$$\mathbf{X} = [x \quad y \quad z]^T$$

Representation Formalism

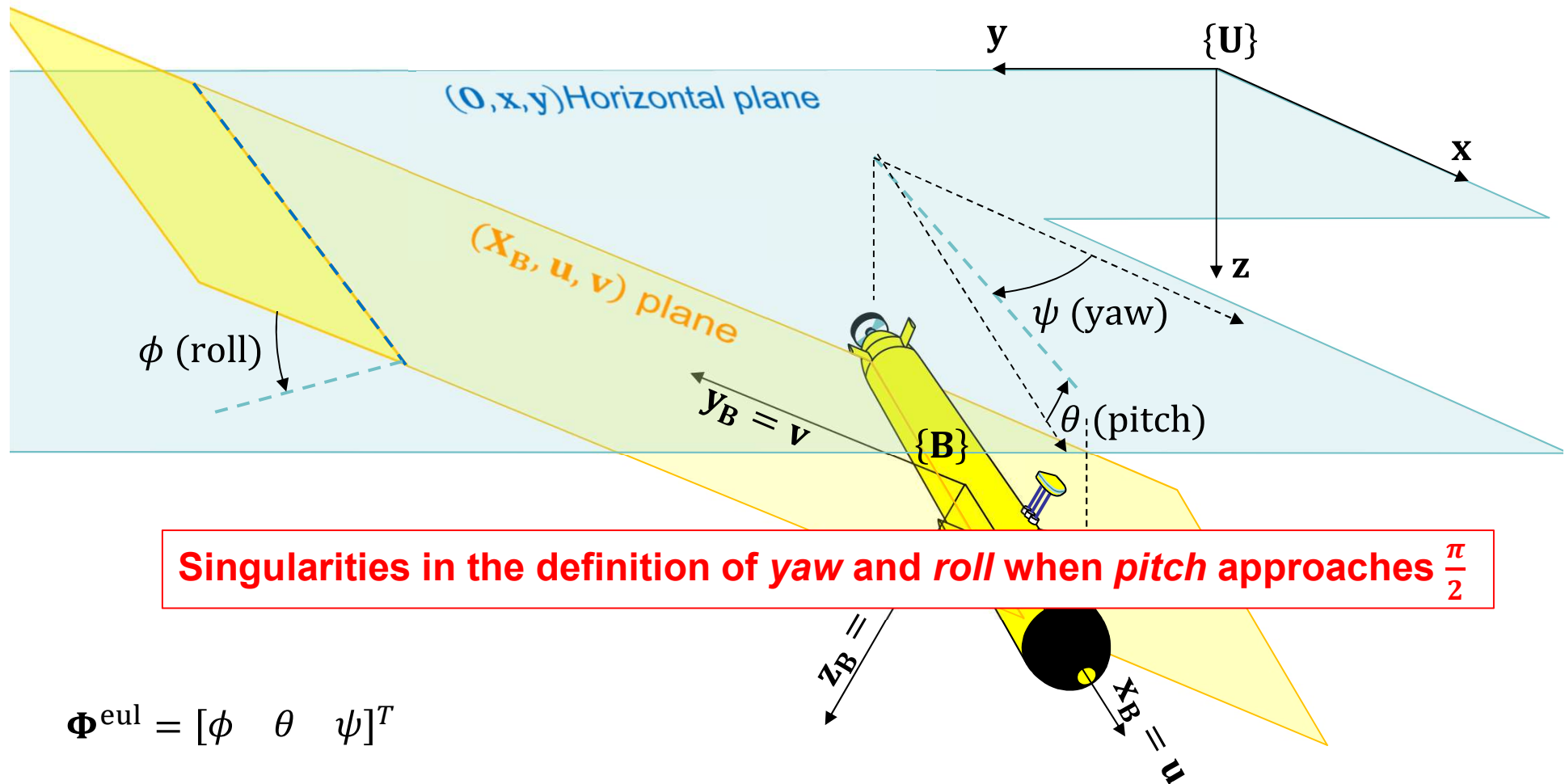
- Attitude, Orientation : Euler angles



$$\Phi^{\text{eul}} = [\phi \quad \theta \quad \psi]^T$$

Representation Formalism

- Attitude, Orientation : Euler angles

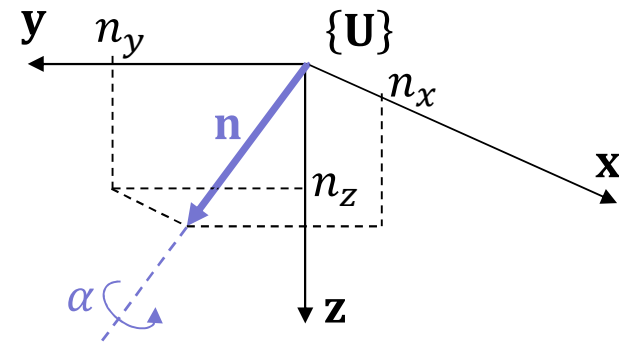
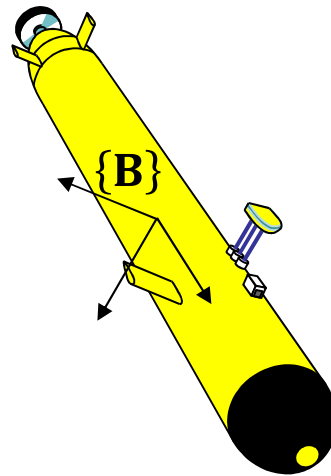


Singularities in the definition of yaw and roll when pitch approaches $\frac{\pi}{2}$

$$\Phi^{\text{eul}} = [\phi \quad \theta \quad \psi]^T$$

Representation Formalism

- Attitude, Orientation : Rotation Matrix



$$\mathbf{n} = [n_x \quad n_y \quad n_z]^T, \|\mathbf{n}\| = 1$$

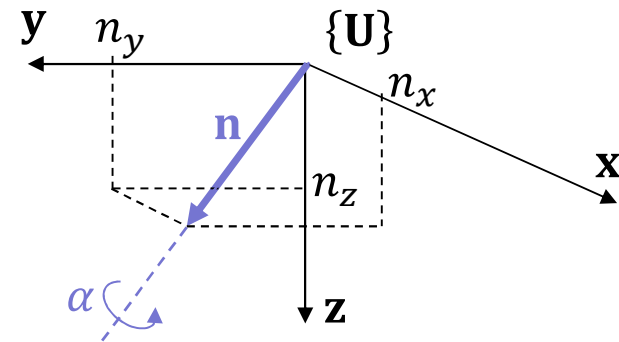
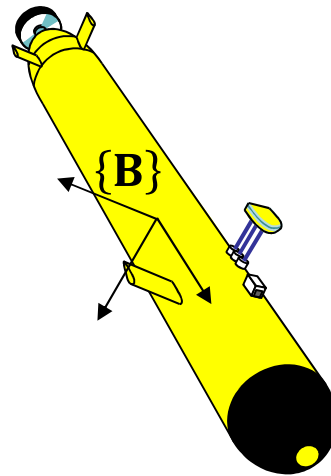
$$\mathbf{R} = \mathbf{P} + \cos \alpha \cdot (\mathbf{I} - \mathbf{P}) + \sin \alpha \cdot \mathbf{Q}$$

$$\text{with : } \mathbf{P} = \mathbf{n} \cdot \mathbf{n}^T, \mathbf{Q} = \Lambda(\mathbf{n})$$

$$\mathbf{Q} = \begin{pmatrix} 0 & -n_z & n_y \\ n_z & 0 & -n_x \\ -n_y & n_x & 0 \end{pmatrix}$$

Representation Formalism

- Attitude, Orientation : Quaternion



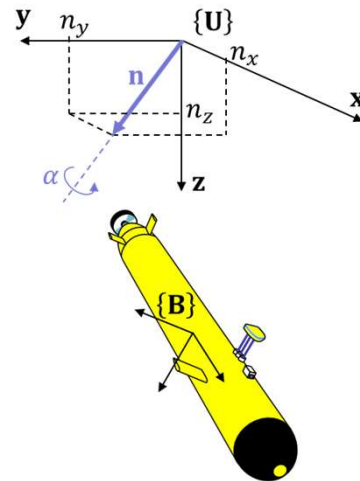
$$\mathbf{n} = [n_x \quad n_y \quad n_z]^T, \|\mathbf{n}\| = 1$$

$$\mathbf{Q}_B = \left[\cos\left(\frac{\alpha}{2}\right) \quad \mathbf{n}^T \cdot \sin\left(\frac{\alpha}{2}\right) \right]^T$$

$$\mathbf{Q}_B = [a \quad b \quad c \quad d]^T$$

Representation Formalism

- Attitude, Orientation : Rot mat. vs Quaternion



$$\mathbf{R} = \mathbf{P} + \cos \alpha \cdot (\mathbf{I} - \mathbf{P}) + \sin \alpha \cdot \mathbf{Q}$$

with : $\mathbf{P} = \mathbf{n} \cdot \mathbf{n}^T$, $\mathbf{Q} = \wedge(\mathbf{n})$

$$\mathbf{Q} = \begin{pmatrix} 0 & -n_z & n_y \\ n_z & 0 & -n_x \\ -n_y & n_x & 0 \end{pmatrix}$$

$$\mathbf{Q}_B = \left[\cos\left(\frac{\alpha}{2}\right) \quad \mathbf{n}^T \cdot \sin\left(\frac{\alpha}{2}\right) \right]^T$$

$$\mathbf{Q}_B = [a \quad b \quad c \quad d]^T$$

$$\mathbf{R} = \begin{pmatrix} a^2 + b^2 - c^2 - d^2 & 2 \cdot b \cdot c - 2 \cdot a \cdot d & 2 \cdot a \cdot c + 2 \cdot b \cdot d \\ 2 \cdot a \cdot d + 2 \cdot b \cdot c & a^2 - b^2 + c^2 - d^2 & 2 \cdot c \cdot d - 2 \cdot a \cdot b \\ 2 \cdot b \cdot d - 2 \cdot a \cdot c & 2 \cdot a \cdot b - 2 \cdot c \cdot d & a^2 - b^2 - c^2 + d^2 \end{pmatrix}$$

$$r = \pm \frac{1}{2} \cdot \sqrt{1 + \text{Tr}(\mathbf{R})}; \begin{cases} a = r & c = \frac{1}{4 \cdot r} \cdot (R_{13} - R_{31}) \\ b = \frac{1}{4 \cdot r} \cdot (R_{32} - R_{23}) & d = \frac{1}{4 \cdot r} \cdot (R_{21} - R_{12}) \end{cases}$$

Quaternion, basic relations

- Any rotation of an angle α around a unitary vector \mathbf{n} can be expressed by the unitary quaternion :

$$\mathbf{Q} = \left[\cos\left(\frac{\alpha}{2}\right) \quad \mathbf{n}^T \cdot \sin\left(\frac{\alpha}{2}\right) \right]^T, \quad \|\mathbf{n}\| = 1 \rightarrow \|\mathbf{Q}\| = 1$$

- The composition of 2 rotations, \mathbf{Q}_1 and \mathbf{Q}_2 can be expressed using the (non commutative) quaternionic multiplication, resulting in the \mathbf{Q}_3 quaternion such that :

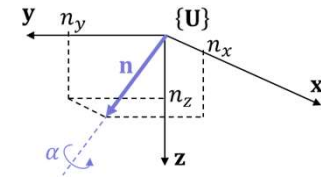
$$\mathbf{Q}_3 = \mathbf{Q}_2 \otimes \mathbf{Q}_1$$

- The conjugate of a quaternion \mathbf{Q} is denoted \mathbf{Q}^* and exhibits the following properties:

$$\mathbf{Q}^* = \left[\cos\left(\frac{\alpha}{2}\right) \quad -\mathbf{n}^T \cdot \sin\left(\frac{\alpha}{2}\right) \right]^T;$$

$$(\mathbf{Q}_1 \otimes \mathbf{Q}_2)^* = \mathbf{Q}_2^* \otimes \mathbf{Q}_1^* ;$$

$$\mathbf{Q} \otimes \mathbf{Q}^* = \|\mathbf{Q}\| \cdot \mathbf{1}_Q, \text{ where } \mathbf{1}_Q = [1, \mathbf{0}, \mathbf{0}, \mathbf{0}]^T \text{ is called } \textit{identity quaternion}$$



- A vector $\mathbf{v} \in \mathbb{R}^3$ can be expressed as a pure imaginary (non unitary) quaternion as:

$$\mathbf{V} = [0, \mathbf{v}^T]^T$$

- The rotation \mathbf{Q} applied on a vector $\mathbf{v}_1 \in \mathbb{R}^3$ results in a vector \mathbf{v}_2 expressed as:

$$\mathbf{V}_2 = \mathbf{Q} \otimes \mathbf{V}_1 \otimes \mathbf{Q}^*, \text{ where } \mathbf{V}_1 = [0, \mathbf{v}_1^T]^T \text{ and } \mathbf{V}_2 = [0, \mathbf{v}_2^T]^T$$

Quaternion, basic relations

- An object, onto which a frame $\{\mathbf{B}\}: (\mathbf{X}_B, \mathbf{u}, \mathbf{v}, \mathbf{w})$ is rigidly attached, in rotation w.r.t an inertial frame $\{\mathbf{U}\}: (\mathbf{O}, \mathbf{x}, \mathbf{y}, \mathbf{z})$, has an angular velocity vector denoted $\boldsymbol{\omega}$. The orientation of $\{\mathbf{B}\}$ w.r.t $\{\mathbf{U}\}$ is denoted with quaternion \mathbf{Q} . Hence the following relations hold :

$$\boldsymbol{\Omega}_B = 2 \cdot \mathbf{Q}^* \otimes \dot{\mathbf{Q}}, \text{ where } \boldsymbol{\Omega}_B = [0, \boldsymbol{\omega}_B^T]^T \text{ and } \boldsymbol{\omega}_B = [p, q, r]^T \text{ is } \boldsymbol{\omega} \text{ expressed in } \{\mathbf{B}\}, \text{ and}$$

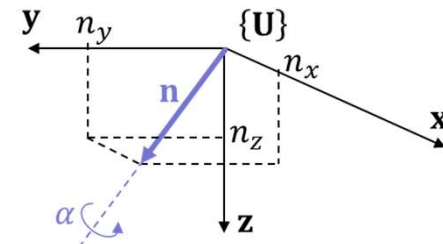
$$\boldsymbol{\Omega}_0 = 2 \cdot \dot{\mathbf{Q}} \otimes \mathbf{Q}^*, \text{ where } \boldsymbol{\Omega}_0 = [0, \boldsymbol{\omega}_0^T]^T \text{ and } \boldsymbol{\omega}_0 \text{ is } \boldsymbol{\omega} \text{ expressed in } \{\mathbf{U}\}$$

- The left-multiplication of previous relation by \mathbf{Q} yields the kinematic rotational model of the moving object as:

$$\dot{\mathbf{Q}} = \frac{1}{2} \cdot \mathbf{Q} \otimes \boldsymbol{\Omega}_B,$$

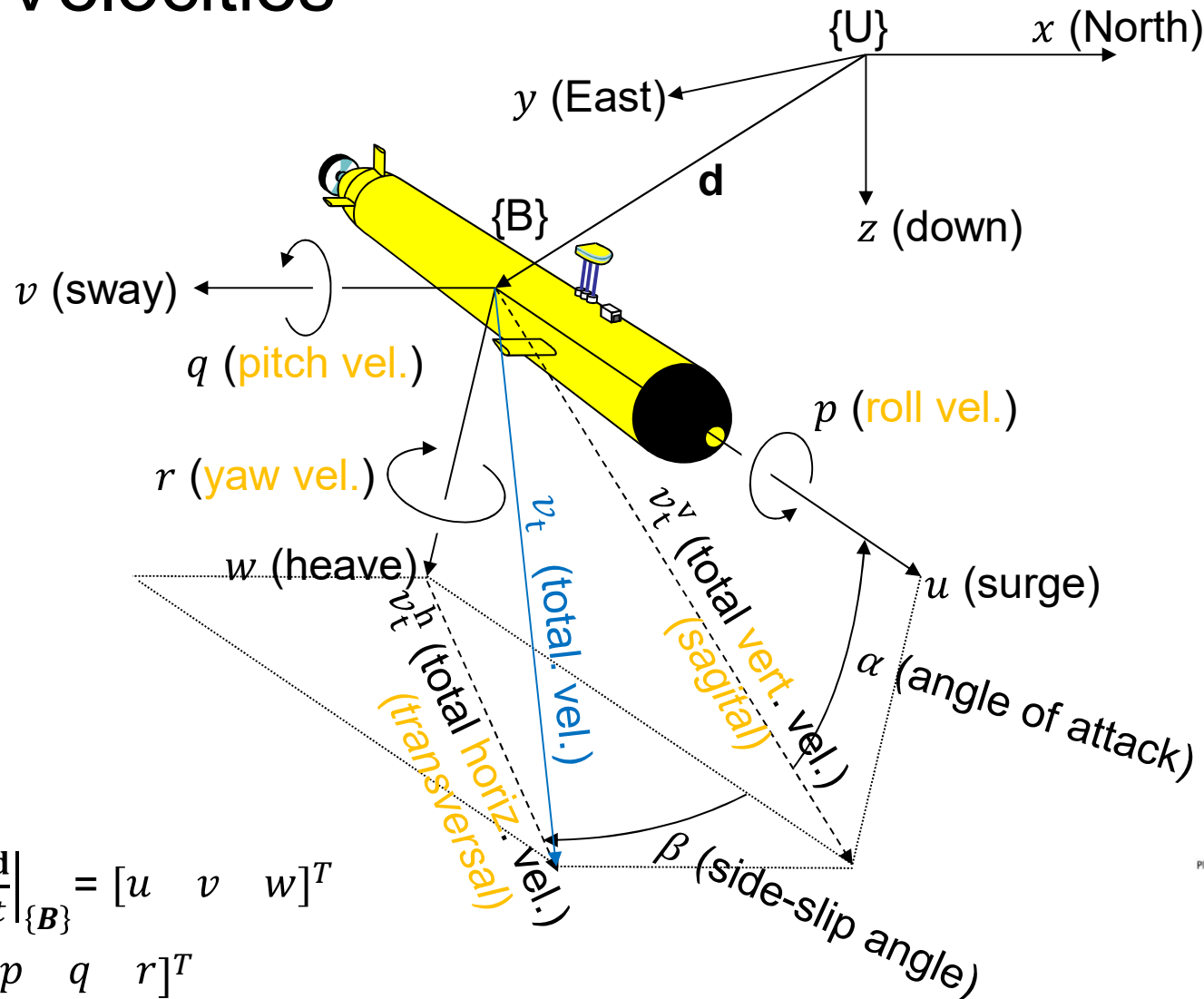
$$\text{where } \boldsymbol{\Omega}_B = [0, \boldsymbol{\omega}_B^T]^T, \boldsymbol{\omega}_B = [p, q, r]^T$$

and p , q and r denote the object rotational velocities expressed in its own $\{\mathbf{B}\}$ frame, as described in the sequel.



Representation Formalism

- Velocities



$$\mathbf{v}_B = \left. \frac{d\mathbf{d}}{dt} \right|_{\{B\}} = [u \quad v \quad w]^T$$

$$\boldsymbol{\omega}_B = [p \quad q \quad r]^T$$

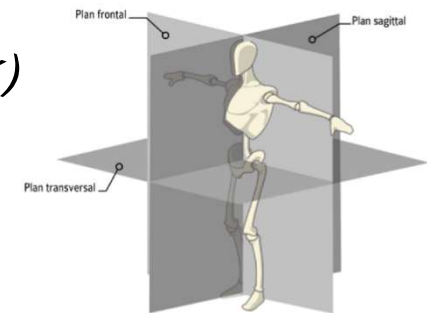


Figure 1 : Plans anatomiques (crédit photo sci-sport.com)

Representation Formalism

	Euler angles	Quaternions	Rotation Matrix
Position	$\mathbf{X} = [x \ y \ z]^T$		
Attitude	$\Phi^{\text{eul}} = [\phi \ \theta \ \psi]^T$	\mathbf{Q}	\mathbf{R}
Velocities	$\mathbf{V}_B = [u \ v \ w]^T, \boldsymbol{\omega}_B = [p \ q \ r]^T$		
State	$\chi = \begin{bmatrix} \boldsymbol{\eta} = \begin{bmatrix} \mathbf{X} \\ \Phi^{\text{eul}} \end{bmatrix} \\ \mathbf{v} = \begin{bmatrix} \mathbf{V}_B \\ \boldsymbol{\omega}_B \end{bmatrix} \end{bmatrix}$	$\chi = \begin{bmatrix} \boldsymbol{\eta} = \begin{bmatrix} \mathbf{X} \\ \mathbf{Q} \end{bmatrix} \\ \mathbf{v} = \begin{bmatrix} \mathbf{V}_B \\ \boldsymbol{\omega}_B \end{bmatrix} \end{bmatrix}$	$\chi = \{\mathbf{X}, \mathbf{R}, \mathbf{V}_B, \boldsymbol{\omega}_B\}$
Kinematic Model $\dot{\boldsymbol{\eta}} = \mathbf{f}(\mathbf{v})$	$\begin{bmatrix} \dot{\mathbf{X}} \\ \dot{\Phi}^{\text{eul}} \end{bmatrix} = \mathbf{R}_{\text{cin}}^{\text{eul}} \cdot \begin{bmatrix} \mathbf{V}_B \\ \boldsymbol{\omega}_B \end{bmatrix}$	$\begin{cases} \dot{\mathbf{X}} = \mathbf{Q} \otimes [0, \mathbf{V}_B^T]^T \otimes \mathbf{Q}^* \\ \dot{\mathbf{Q}} = \frac{1}{2} \cdot \mathbf{Q} \otimes \underbrace{[0, \boldsymbol{\omega}_B^T]^T}_{\boldsymbol{\Omega}_B} \end{cases}$	$\begin{cases} \dot{\mathbf{X}} = \mathbf{R} \cdot \mathbf{V}_B \\ \dot{\mathbf{R}} = \mathbf{R} \cdot (\boldsymbol{\omega}_B \wedge) \end{cases}$

$$\mathbf{R}_{\text{cin}}^{\text{eul}} = \begin{bmatrix} \cos \psi \cdot \cos \theta & \cos \psi \cdot \sin \theta \cdot \sin \phi - \sin \psi \cdot \cos \phi & \cos \psi \cdot \sin \theta \cdot \cos \phi + \sin \psi \cdot \sin \phi & 0 & 0 & 0 \\ \sin \psi \cdot \cos \theta & \sin \psi \cdot \sin \theta \cdot \sin \phi + \cos \psi \cdot \cos \phi & \sin \psi \cdot \sin \theta \cdot \cos \phi - \cos \psi \cdot \sin \phi & 0 & 0 & 0 \\ -\sin \theta & \cos \theta \cdot \sin \phi & \cos \theta \cdot \cos \phi & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & \sin \phi \cdot \tan \theta & \cos \phi \cdot \tan \theta \\ 0 & 0 & 0 & 0 & \cos \phi & -\sin \phi \\ 0 & 0 & 0 & 0 & \frac{\sin \phi}{\cos \theta} & \frac{\cos \phi}{\cos \theta} \end{bmatrix}$$

Gimbal lock, if $\theta = \pm \frac{\pi}{2}$

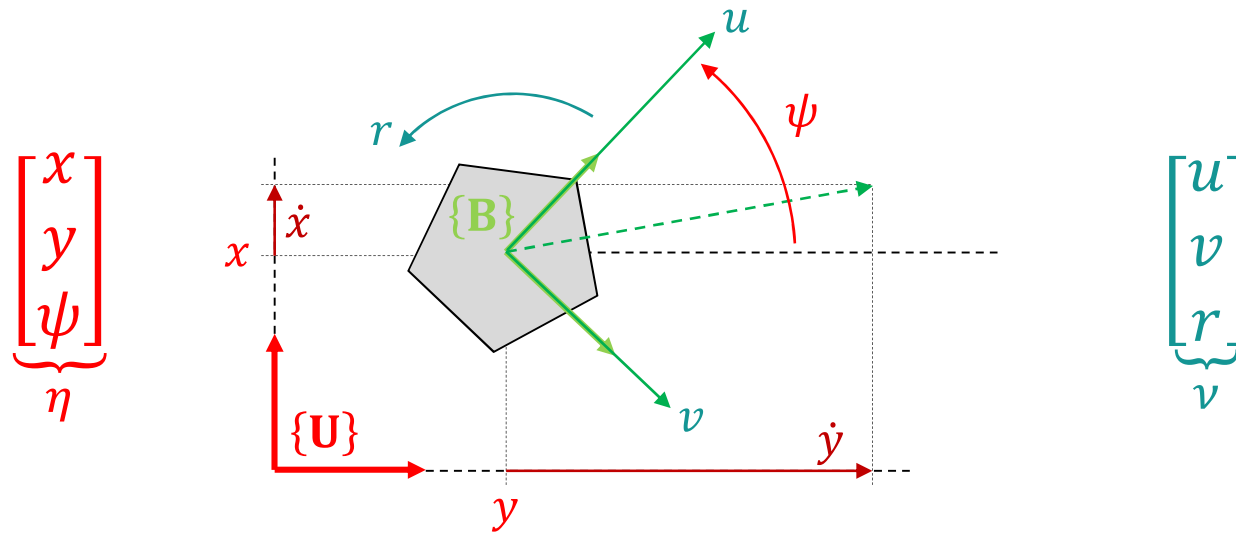
$$\boldsymbol{\omega}_B \wedge = \begin{pmatrix} 0 & -w_z & w_y \\ w_z & 0 & -w_x \\ -w_y & w_x & 0 \end{pmatrix}$$

II – Models

b) Kinematic Model

Kinematic Model

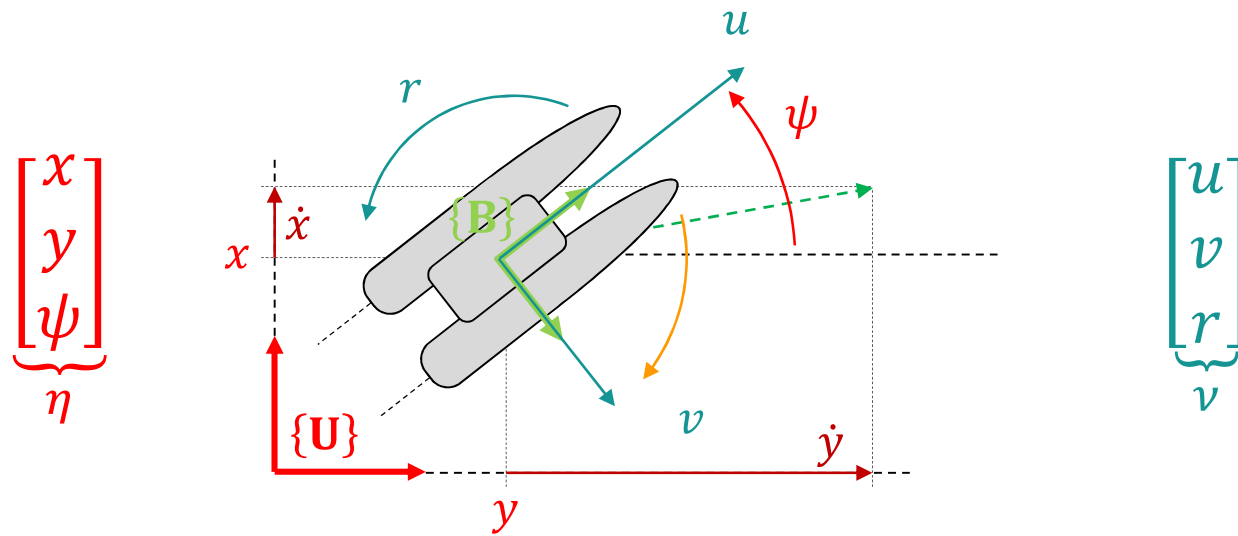
- 2D, Cartesian, no constraint



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\boldsymbol{\eta}}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}} \rightarrow \dot{\boldsymbol{\eta}} = \mathbf{R} \cdot \mathbf{v}$$

Kinematic Model

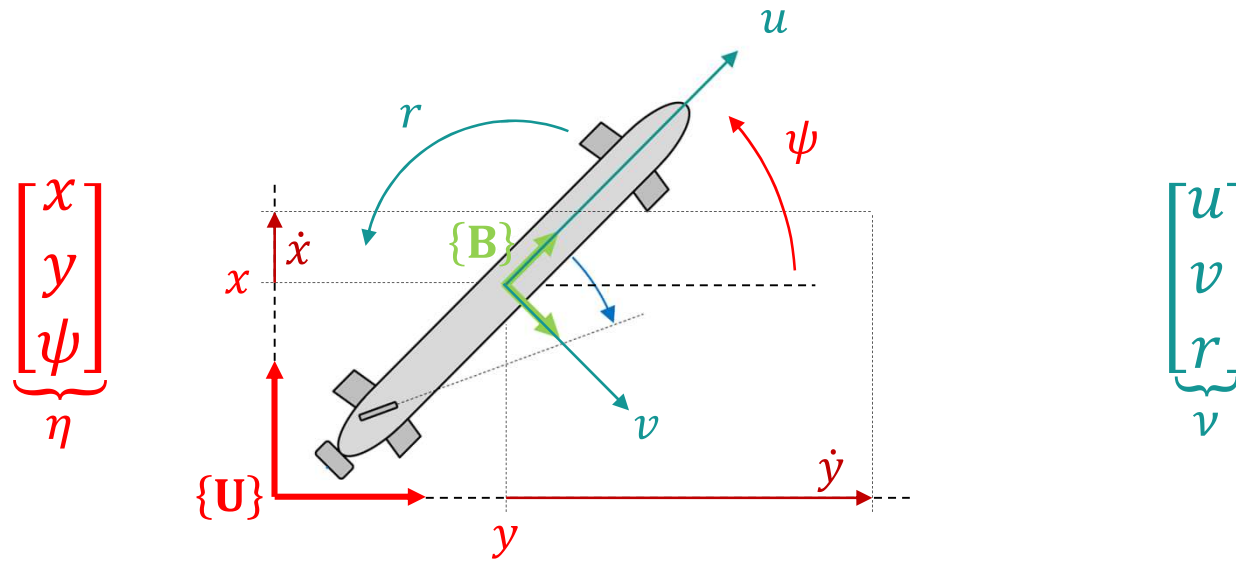
- 2D, Surface Craft, Cartesian, no constraint



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\boldsymbol{\eta}}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}} \rightarrow \dot{\boldsymbol{\eta}} = \mathbf{R} \cdot \mathbf{v}$$

Kinematic Model

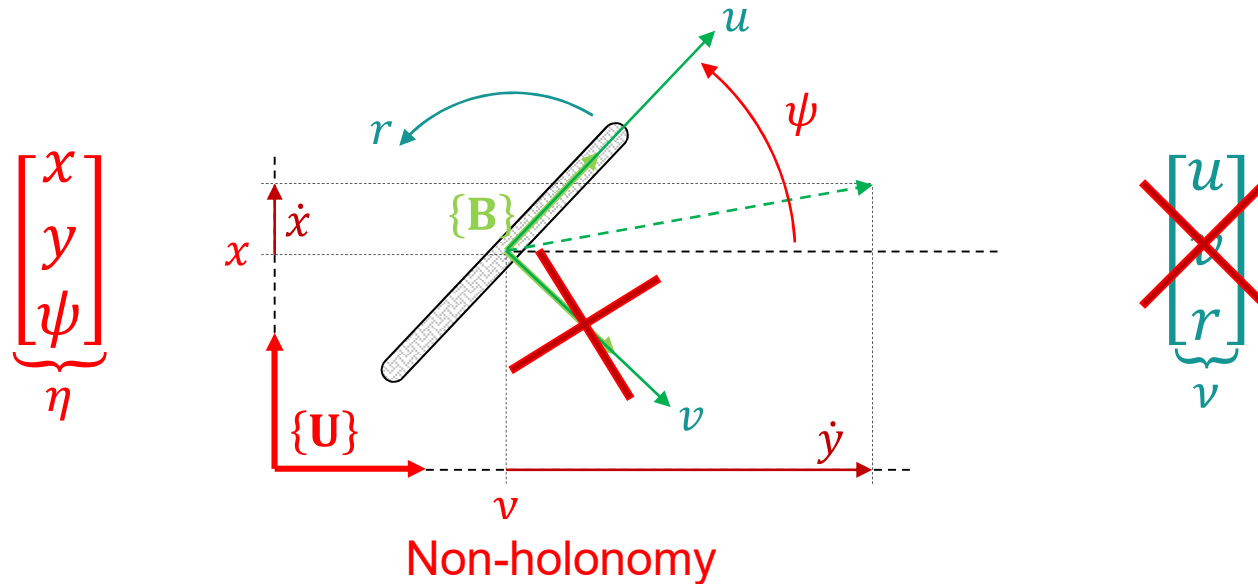
- 2D, Surface Craft, Cartesian, no constraint



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}} \rightarrow \dot{\eta} = \mathbf{R} \cdot \mathbf{v}$$

Kinematic Model

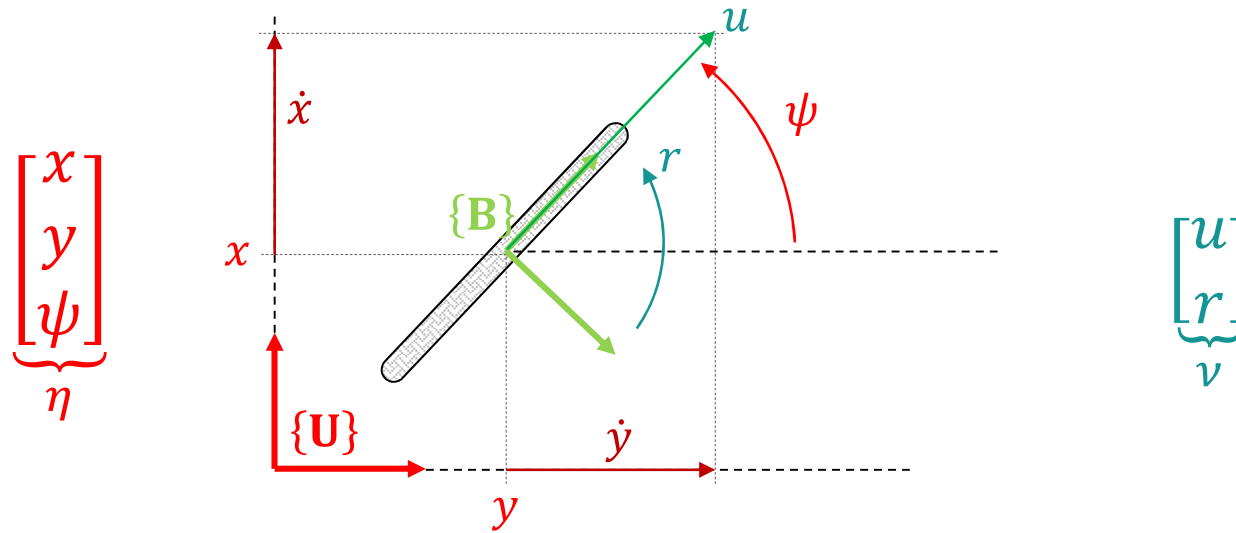
- 2D, Cartesian, the Wheel, nonholonomic constraint : $v = \dot{x} \cdot \cos \psi + \dot{y} \cdot \sin \psi = 0$



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ \psi \\ r \end{bmatrix}}_{\mathbf{v}} \rightarrow \dot{\eta} = \mathbf{R} \cdot \mathbf{v}$$

Kinematic Model

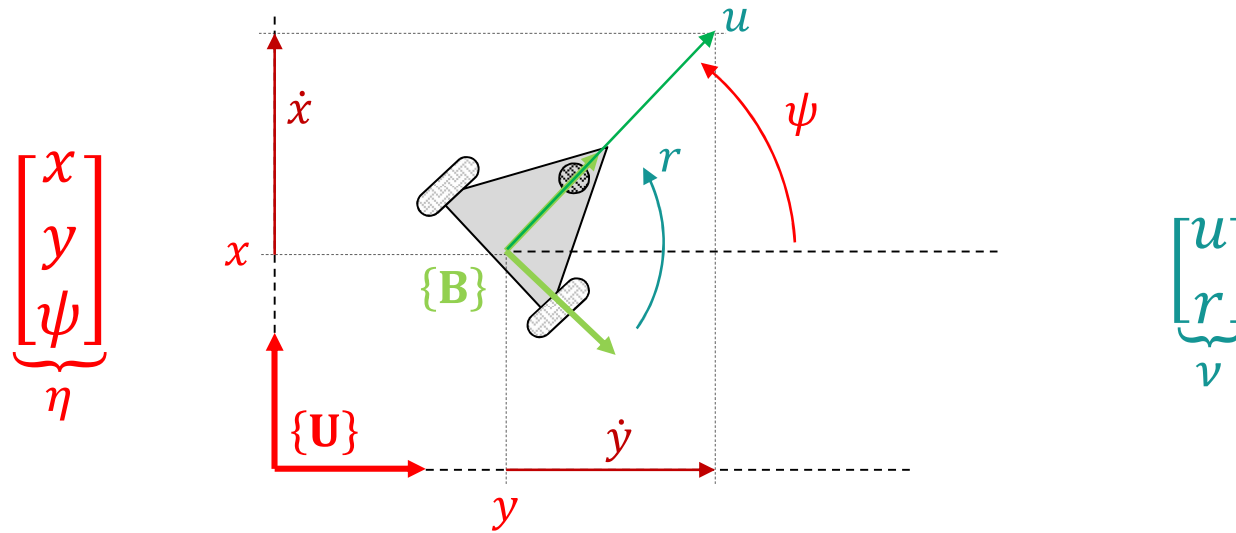
- 2D, Cartesian, the Wheel, nonholonomic constraint : $v = \dot{x} \cdot \cos \psi + \dot{y} \cdot \sin \psi = 0$



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & 0 \\ \sin \psi & 0 \\ 0 & 1 \end{bmatrix}}_{J} \cdot \underbrace{\begin{bmatrix} u \\ r \end{bmatrix}}_v \rightarrow \dot{\eta} = J \cdot v$$

Kinematic Model

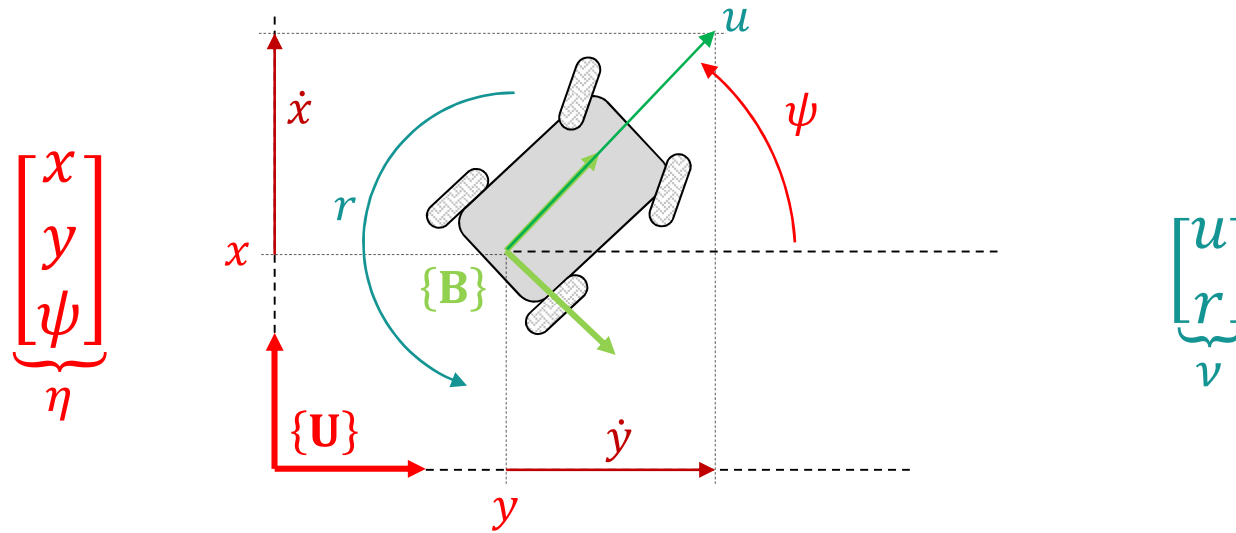
- 2D, Cartesian, the Unicycle, nonholonomic constraint : $v = \dot{x} \cdot \cos \psi + \dot{y} \cdot \sin \psi = 0$



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & 0 \\ \sin \psi & 0 \\ 0 & 1 \end{bmatrix}}_{J} \cdot \underbrace{\begin{bmatrix} u \\ r \end{bmatrix}}_{\mathbf{v}} \rightarrow \dot{\eta} = J \cdot \mathbf{v}$$

Kinematic Model

- 2D, Cartesian, the Car, nonholonomic constraint : $v = \dot{x} \cdot \cos \psi + \dot{y} \cdot \sin \psi = 0$



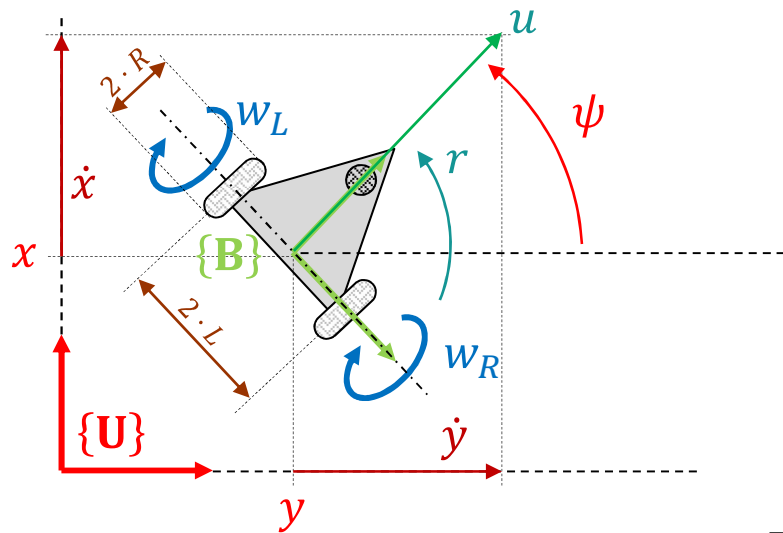
$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\boldsymbol{\eta}}} = \underbrace{\begin{bmatrix} \cos \psi & 0 \\ \sin \psi & 0 \\ 0 & 1 \end{bmatrix}}_{\mathbf{J}} \cdot \underbrace{\begin{bmatrix} u \\ r \end{bmatrix}}_{\mathbf{v}} \rightarrow \dot{\boldsymbol{\eta}} = \mathbf{J} \cdot \mathbf{v}$$

II – Models

c) Actuation Model

Actuation Model

- 2D, the Unicycle



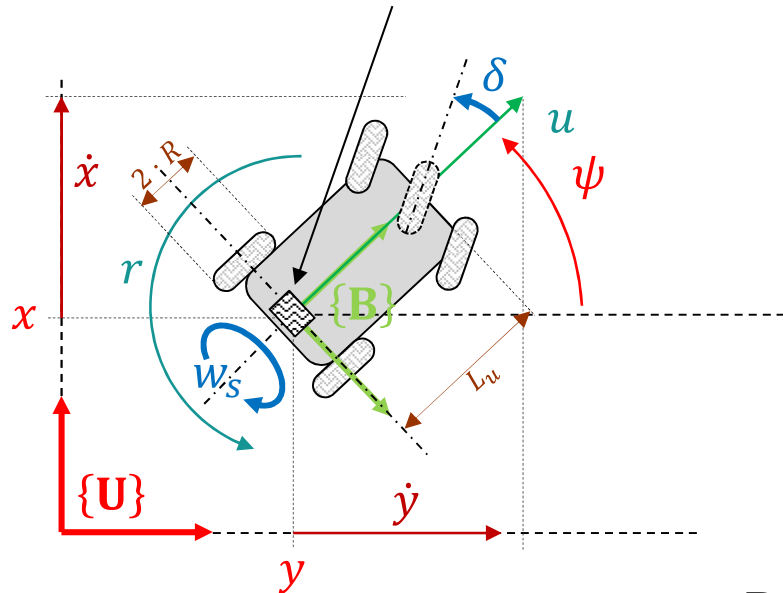
$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\hat{q}} = \underbrace{\begin{bmatrix} \cos \psi & 0 \\ \sin \psi & 0 \\ 0 & 1 \end{bmatrix}}_J \cdot \underbrace{\begin{bmatrix} u \\ r \end{bmatrix}}_{\mathbf{v}}$$

$$\underbrace{\begin{bmatrix} u \\ r \end{bmatrix}}_{\mathbf{v}} = \underbrace{\begin{bmatrix} \frac{R}{2} & \frac{R}{2} \\ R & -R \\ \frac{1}{2 \cdot L} & \frac{1}{2 \cdot L} \end{bmatrix}}_A \cdot \underbrace{\begin{bmatrix} w_R \\ w_L \end{bmatrix}}_{\mathbf{U}} \rightarrow \mathbf{v} = \mathbf{A} \cdot \mathbf{U}$$

Actuation Model

- 2D, the Car

differential drive (K_R)

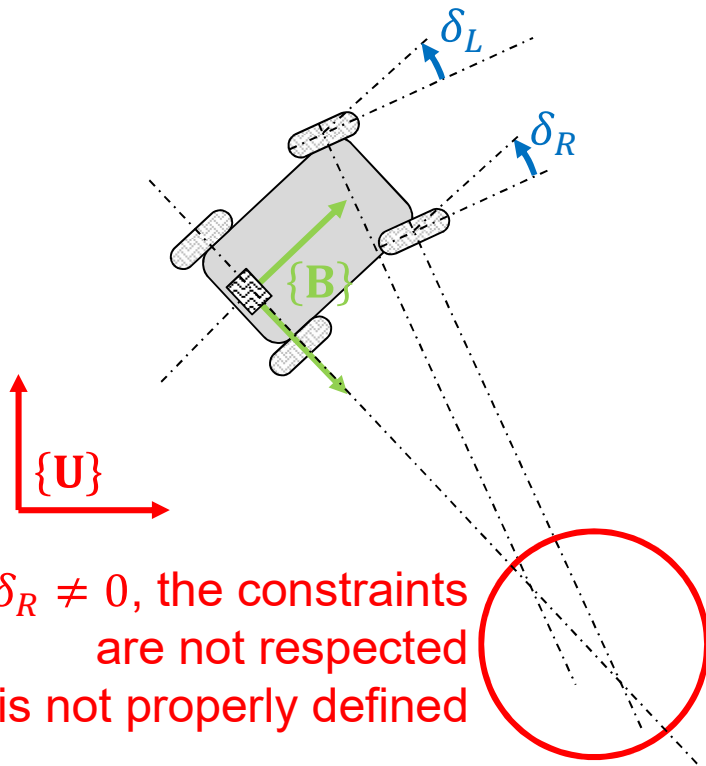
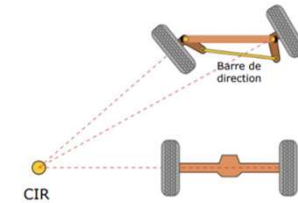


$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & 0 \\ \sin \psi & 0 \\ 0 & 1 \end{bmatrix}}_J \cdot \underbrace{\begin{bmatrix} u \\ r \\ v \end{bmatrix}}_v$$

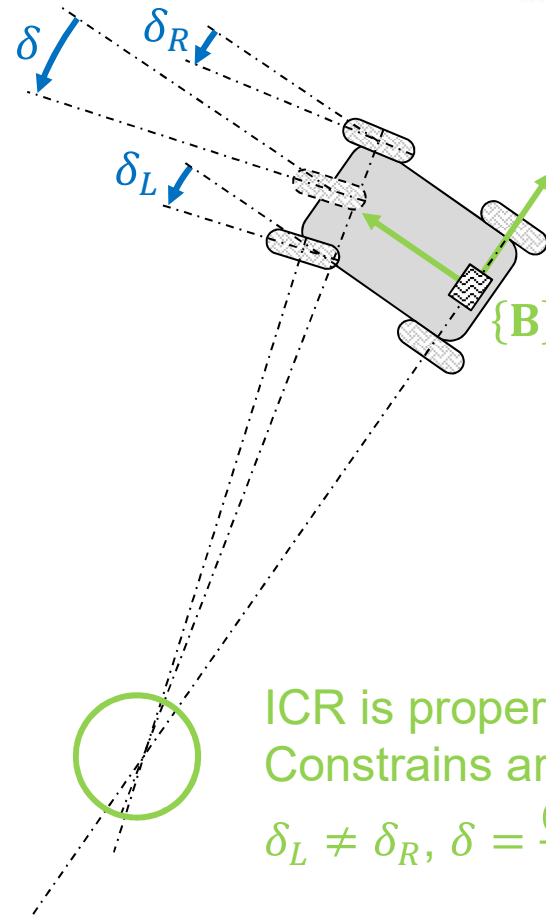
$$\underbrace{\begin{bmatrix} u \\ r \\ v \end{bmatrix}}_v = \begin{bmatrix} \frac{R}{K_R} \cdot \omega_s \\ u \cdot \tan \delta \\ L_u \end{bmatrix} \rightarrow v = A(U)$$

Actuation Model

- 2D, the Car, non-sliding constraints



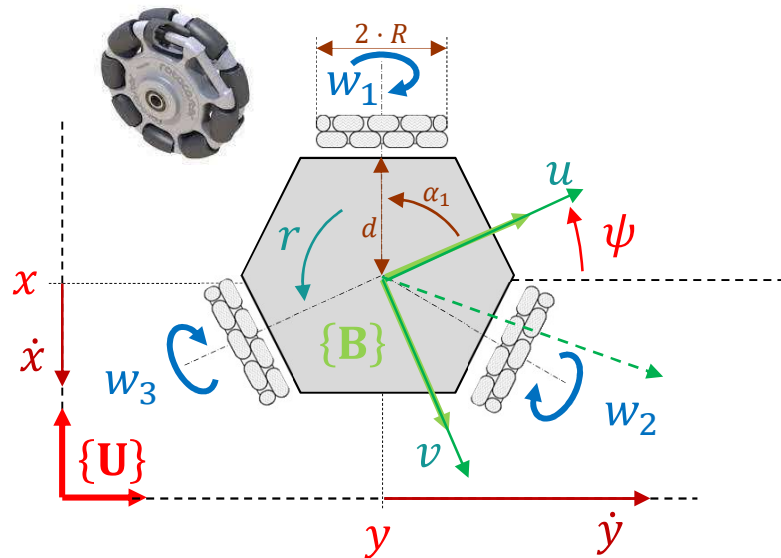
If $\delta_L = \delta_R \neq 0$, the constraints are not respected
ICR is not properly defined



ICR is properly defined
Constraints are respected
 $\delta_L \neq \delta_R, \delta = \frac{(\delta_L + \delta_R)}{2}$

Actuation Model

- 2D, Cartesian, Omni-directional *sweedish wheels* system



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

$$\rightarrow \dot{\eta} = \mathbf{R} \cdot \mathbf{v}$$

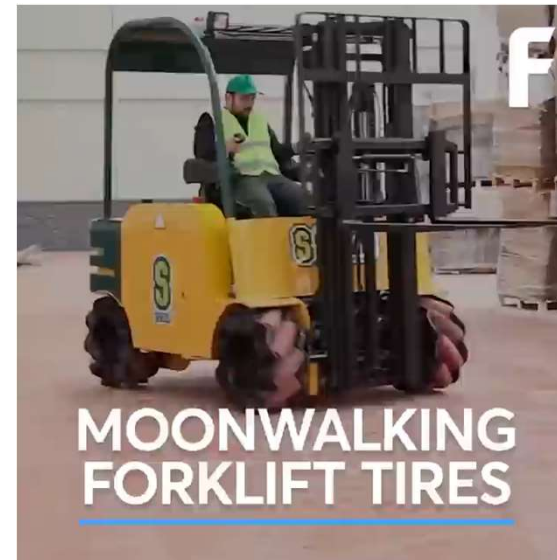
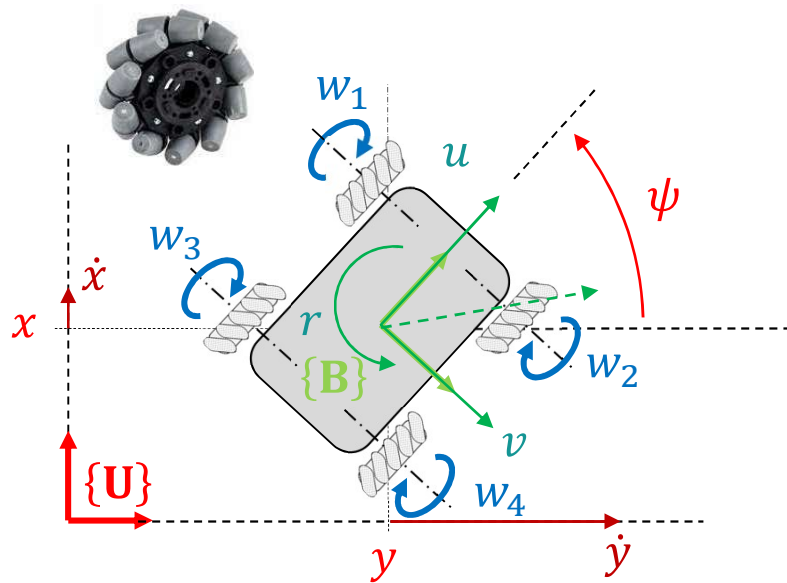
$$\underbrace{\begin{bmatrix} w_1 \\ w_2 \\ w_3 \end{bmatrix}}_{\mathbf{U}} = -\frac{1}{R} \cdot \underbrace{\begin{bmatrix} -\sin \alpha_1 & \cos \alpha_1 & d \\ -\sin \alpha_2 & \cos \alpha_2 & d \\ -\sin \alpha_3 & \cos \alpha_3 & d \end{bmatrix}}_{\mathbf{A}^{-1}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

$$\rightarrow \mathbf{v} = \mathbf{A} \cdot \mathbf{U}$$

$$\alpha_1 = \frac{\pi}{3}, \alpha_2 = -\frac{\pi}{3}, \alpha_3 = \pi$$

Actuation Model

- 2D, Cartesian, Omni_directional *Mecanum wheels* system



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

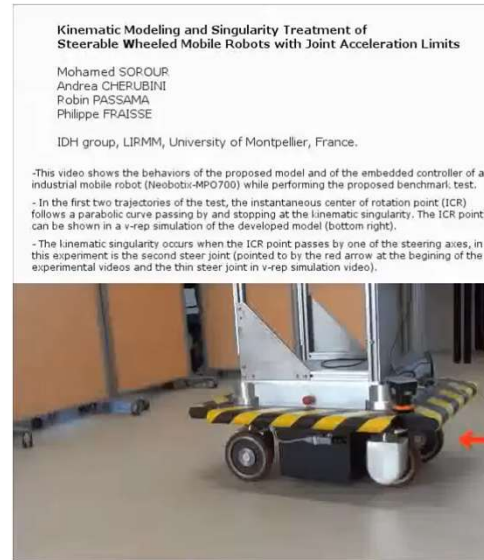
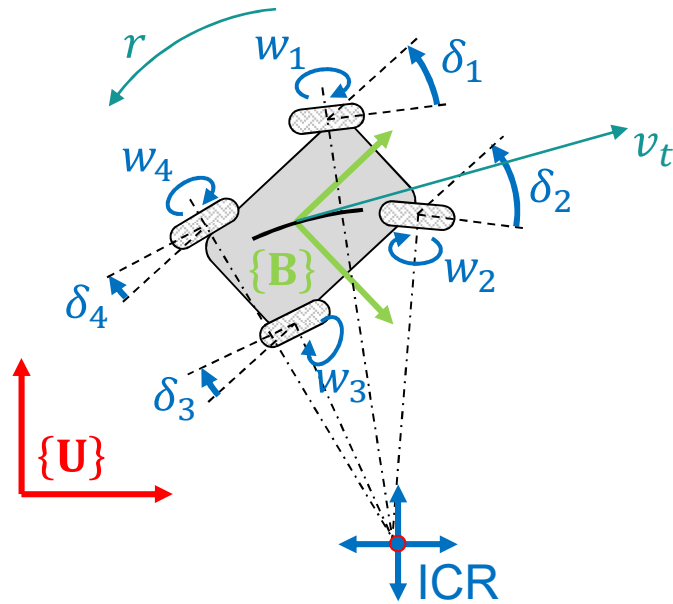
$\rightarrow \dot{\eta} = \mathbf{R} \cdot \mathbf{v}$

$$\underbrace{\begin{bmatrix} w_1 \\ w_2 \\ w_3 \\ w_4 \end{bmatrix}}_{\mathbf{U}} = \underbrace{\begin{bmatrix} 1 & 1 & -(L_1 + L_2) \\ -1 & 1 & -(L_1 + L_2) \\ 1 & -1 & -(L_1 + L_2) \\ -1 & -1 & -(L_1 + L_2) \end{bmatrix}}_{\mathbf{A}^{-1}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

$\rightarrow \mathbf{v} = \mathbf{A} \cdot \mathbf{U}$

Actuation Model

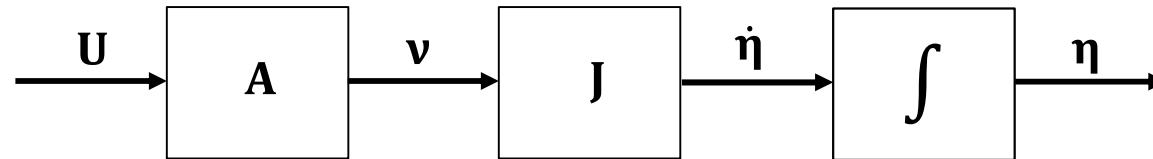
- 2D, the Car, Pseudo-Omni-Directional Wheeled Robots



$$\begin{cases} \mathbf{W} = [w_1 & w_2 & w_3 & w_4]^T \\ \mathbf{\Delta} = [\delta_1 & \delta_2 & \delta_3 & \delta_4]^T \end{cases}, \begin{cases} \mathbf{W} = \mathbf{f}(v_t, r), \text{ subject to } \Phi_{\mathbf{W}}(\mathbf{W}) = 0 \\ \mathbf{\Delta} = \mathbf{g}(v_t, r), \text{ subject to } \Phi_{\mathbf{\Delta}}(\mathbf{\Delta}) = 0 \end{cases}, \left\| \mathbf{X}_{ICR} \Big|_{\{B\}} \right\|^{-1} = \frac{r}{v_t}$$

Simulation

- Kinematic simulation



Video sim Unicycle

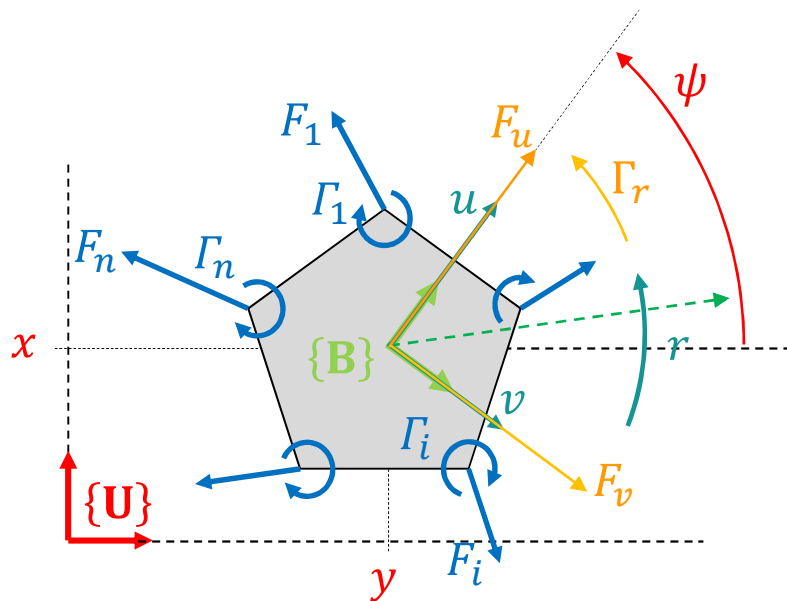
Video sim Omni 3 roues

Video sim Car

Video sim Omni Mecanum

Actuation Model

- 2D, Cartesian, Propulsive actuation



$$\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}_{\dot{\eta}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

$$\rightarrow \dot{\eta} = \mathbf{R}(\eta) \cdot \mathbf{v}$$

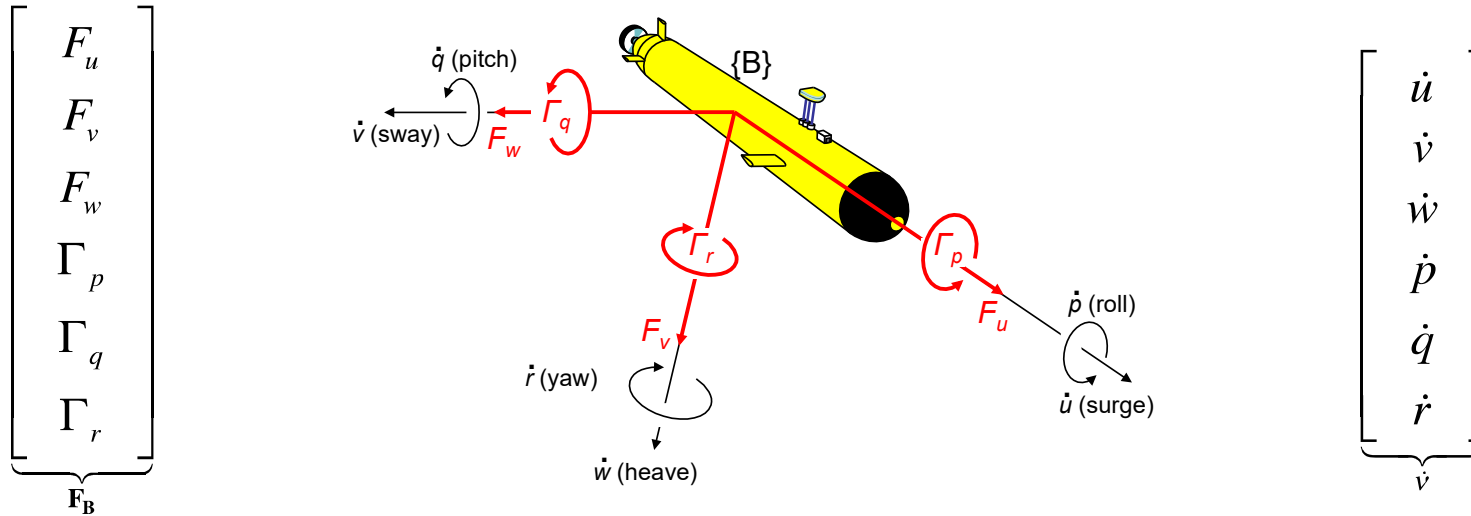
Dynamic Model:

$$\dot{\mathbf{v}} = \mathbf{f}_D(\Theta, \eta, \mathbf{v}, \mathbf{F}_B)$$

$$\underbrace{\begin{bmatrix} F_u \\ F_v \\ \Gamma_r \end{bmatrix}}_{\mathbf{F}_B} = \mathbf{A} \cdot \underbrace{[F_1, \Gamma_1, \dots, F_i, \Gamma_i, \dots, F_n, \Gamma_n]^T}_{\mathbf{F}_m}$$

$$\rightarrow \mathbf{F}_B = \mathbf{A} \cdot \mathbf{F}_m$$

Dynamic Model

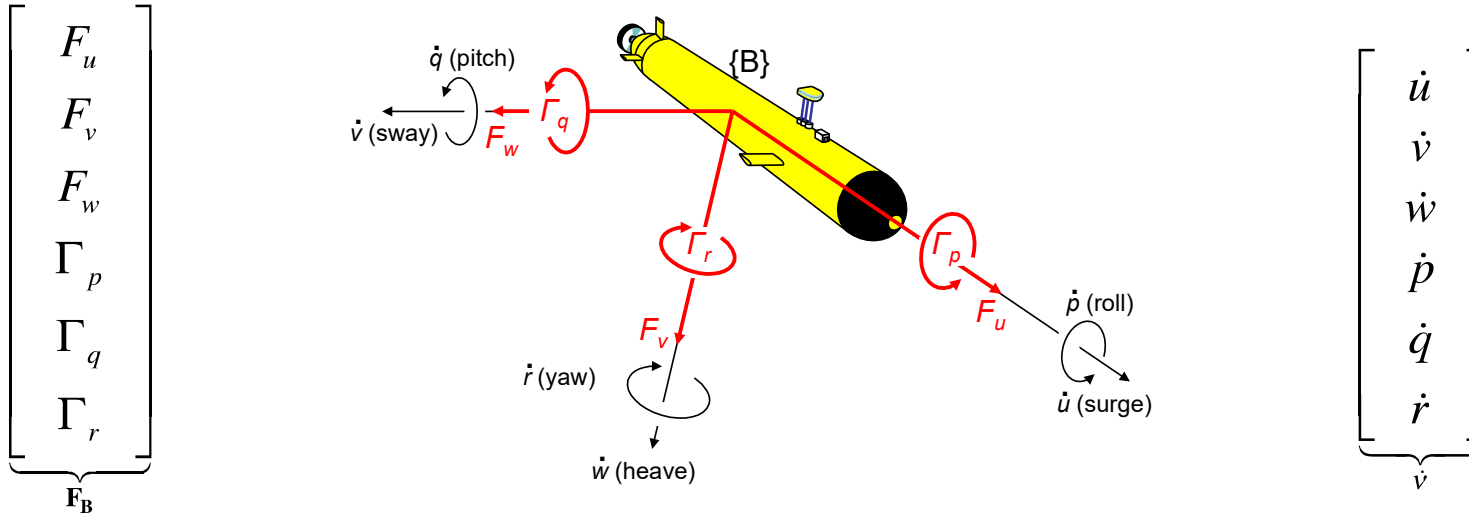


Hydro-dynamic phenomena (fluid/structure interaction) :

- Buoyancy
- Lift and Drag
- Added Mass

$$\mathbf{F}_B = \mathbf{f}_D(\boldsymbol{\Theta}, \mathbf{v}, \boldsymbol{\eta}, \dot{\mathbf{v}})$$

Dynamic Model



$$F_u = X_{\dot{u}} \cdot \dot{u} + X_{u \cdot |u|} \cdot u \cdot |u| + X_{w \cdot q} \cdot w \cdot q + X_{v \cdot r} \cdot v \cdot r + X_G(\eta)$$

$$F_v = Y_{\dot{v}} \cdot \dot{v} + Y_{v \cdot |v|} \cdot v \cdot |v| + Y_{u \cdot r} \cdot u \cdot r + Y_{w \cdot p} \cdot w \cdot p + Y_G(\eta)$$

$$F_w = Z_{\dot{w}} \cdot \dot{w} + Z_{w \cdot |w|} \cdot w \cdot |w| + Z_{u \cdot q} \cdot u \cdot q + Z_{v \cdot p} \cdot v \cdot p + Z_G(\eta)$$

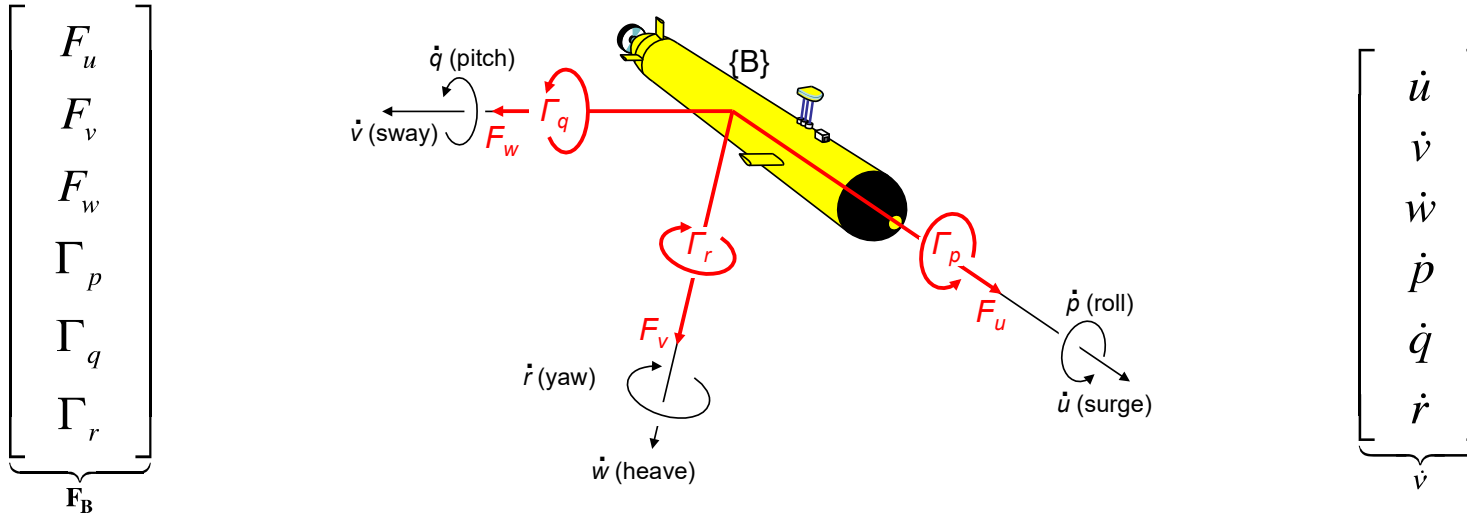
$$\Gamma_p = K_{\dot{p}} \cdot \dot{p} + K_{p \cdot |p|} \cdot p \cdot |p| + K_{q \cdot r} \cdot q \cdot r + K_v \cdot v + K_w \cdot w + K_G(\eta)$$

$$\Gamma_q = M_{\dot{q}} \cdot \dot{q} + M_{q \cdot |q|} \cdot q \cdot |q| + M_{p \cdot r} \cdot p \cdot r + M_u \cdot u + M_w \cdot w + M_G(\eta)$$

$$\Gamma_r = N_{\dot{r}} \cdot \dot{r} + N_{r \cdot |r|} \cdot r \cdot |r| + N_{p \cdot q} \cdot p \cdot q + N_u \cdot u + N_v \cdot v + N_G(\eta)$$

$$\mathbf{F}_B = \mathbf{f}_D(\Theta, \mathbf{v}, \eta, \dot{\mathbf{v}})$$

Dynamic Model



$$F_u = X_{\dot{u}} \cdot \dot{u} + X_{u \cdot |u|} \cdot u \cdot |u| + X_{w \cdot q} \cdot w \cdot q + X_{v \cdot r} \cdot v \cdot r + X_G(\eta)$$

$$F_v = Y_{\dot{v}} \cdot \dot{v} + Y_{v \cdot |v|} \cdot v \cdot |v| + Y_{u \cdot r} \cdot u \cdot r + Y_{w \cdot p} \cdot w \cdot p + Y_G(\eta)$$

$$F_w = Z_{\dot{w}} \cdot \dot{w} + Z_{w \cdot |w|} \cdot w \cdot |w| + Z_{u \cdot q} \cdot u \cdot q + Z_{v \cdot p} \cdot v \cdot p + Z_G(\eta)$$

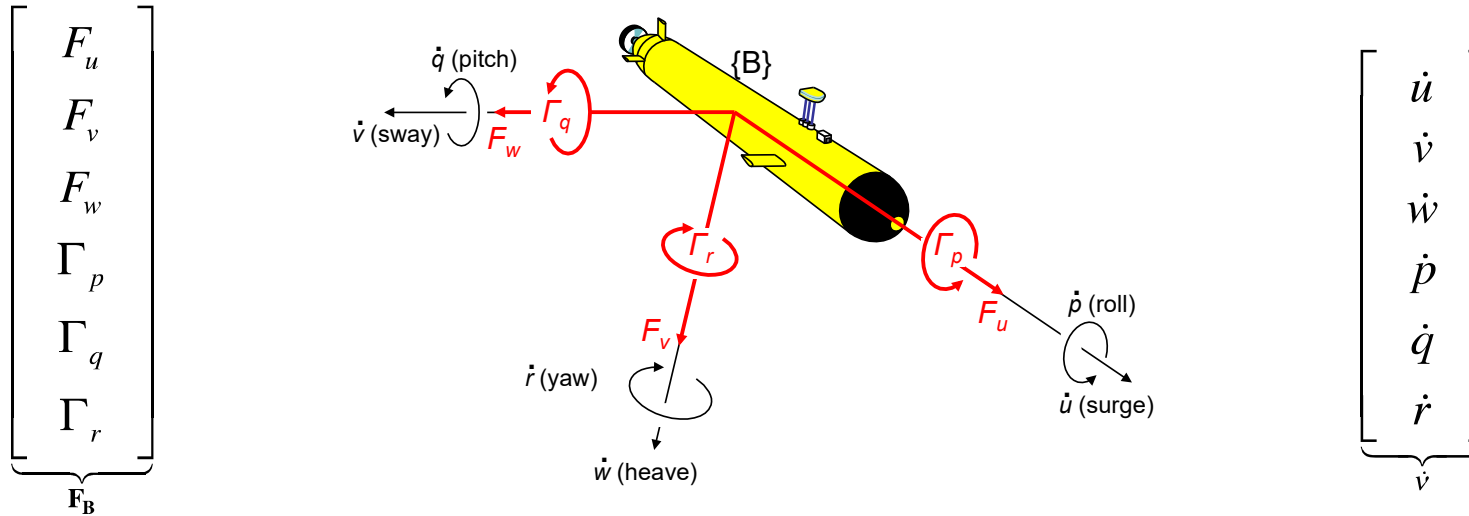
$$\Gamma_p = K_{\dot{p}} \cdot \dot{p} + K_{p \cdot |p|} \cdot p \cdot |p| + K_{q \cdot r} \cdot q \cdot r + K_v \cdot v + K_w \cdot w + K_G(\eta)$$

$$\Gamma_q = M_{\dot{q}} \cdot \dot{q} + M_{q \cdot |q|} \cdot q \cdot |q| + M_{p \cdot r} \cdot p \cdot r + M_u \cdot u + M_w \cdot w + M_G(\eta)$$

$$\Gamma_r = N_{\dot{r}} \cdot \dot{r} + N_{r \cdot |r|} \cdot r \cdot |r| + N_{p \cdot q} \cdot p \cdot q + N_u \cdot u + N_v \cdot v + N_G(\eta)$$

$$\mathbf{F}_B = \mathbf{f}_D(\Theta, \mathbf{v}, \eta, \dot{\mathbf{v}})$$

Dynamic Model



A NONLINEAR UNIFIED STATE-SPACE MODEL FOR SHIP MANEUVERING AND CONTROL IN A SEAWAY

THOR I. FOSSEN

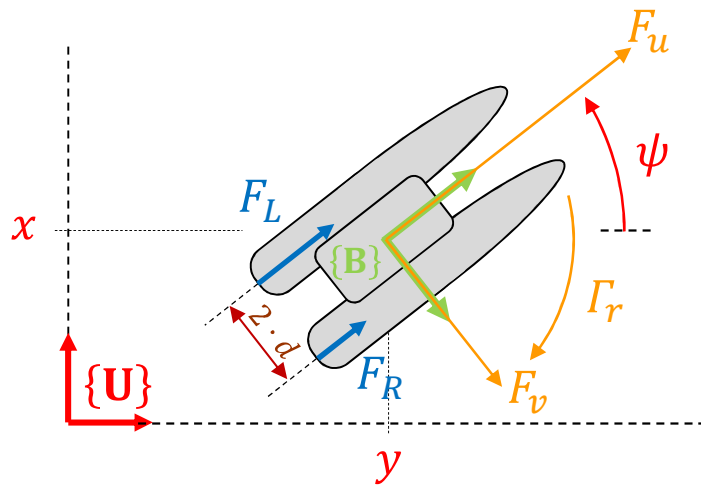
*Department of Engineering Cybernetics
Norwegian University of Science and Technology
NO-7491 Trondheim, Norway
E-mail: fossen@ieee.org*

This article presents a unified state-space model for ship maneuvering, station-keeping, and control in a seaway. The frequency-dependent potential and viscous damping terms, which in classic theory results in a convolution integral not suited for real-time simulation, is compactly represented by using a state-space formulation. The separation of the vessel model into a low-frequency model (represented by zero-frequency added mass and damping) and a wave-frequency model (represented by motion transfer functions or RAOs), which is commonly used for simulation, is hence made superfluous.

Keywords: ship modelling, equations of motion, hydrodynamics, maneuvering, seakeeping, autopilots, dynamic positioning.

Actuation Model

- 2D, Cartesian, Propulsive actuation, ASV



$$\underbrace{\begin{bmatrix} F_u \\ F_v \\ \Gamma_r \end{bmatrix}}_{\mathbf{F}_B} = \underbrace{\begin{bmatrix} 1 & 1 \\ 0 & 0 \\ d & -d \end{bmatrix}}_{\mathbf{A}} \cdot \underbrace{\begin{bmatrix} F_L \\ F_R \end{bmatrix}^T}_{\mathbf{F}_m}$$

$$\rightarrow \mathbf{F}_B = \mathbf{A} \cdot \mathbf{F}_m$$

$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\boldsymbol{\eta}}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

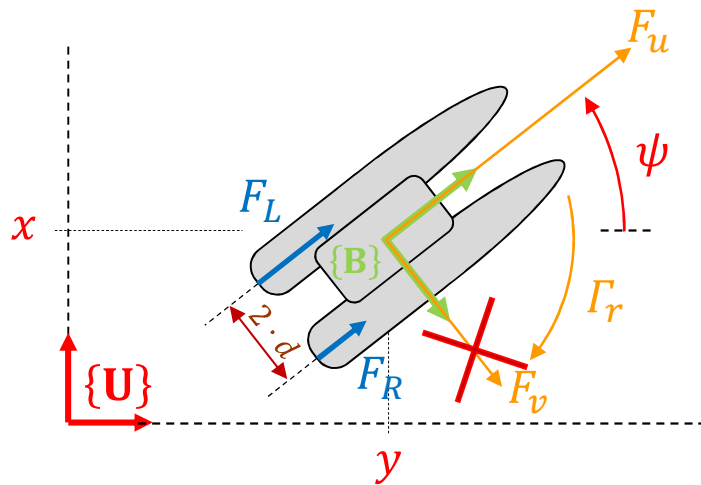
$$\rightarrow \dot{\boldsymbol{\eta}} = \mathbf{R} \cdot \mathbf{v}$$

$$\underbrace{\begin{bmatrix} \dot{u} \\ \dot{v} \\ \dot{r} \end{bmatrix}}_{\dot{\mathbf{v}}} = \begin{cases} \frac{1}{X_{\dot{u}}} \cdot (F_u - X_u \cdot u \cdot |u|) \\ \frac{1}{Y_{\dot{v}}} \cdot (F_v - Y_v \cdot v \cdot |v| - Y_{ur} \cdot u \cdot r) \\ \frac{1}{N_{\dot{u}}} \cdot (\Gamma_r - N_r \cdot r \cdot |r|) \end{cases}$$

$$\rightarrow \dot{\mathbf{v}} = \mathbf{f}_D(\boldsymbol{\Theta}, \mathbf{v}, \boldsymbol{\eta}, \mathbf{F}_B)$$

Actuation Model

- 2D, Cartesian, Propulsive actuation, ASV



$$\begin{aligned}
 \begin{bmatrix} F_u \\ F_v \\ F_r \end{bmatrix}_{\mathbf{F}_B} &= \begin{bmatrix} 1 & 1 \\ 0 & 0 \\ d & -d \end{bmatrix} \cdot \begin{bmatrix} F_L \\ F_R \end{bmatrix}_{\mathbf{F}_m}^T \\
 &\rightarrow \mathbf{F}_B = \mathbf{A} \cdot \mathbf{F}_m
 \end{aligned}$$

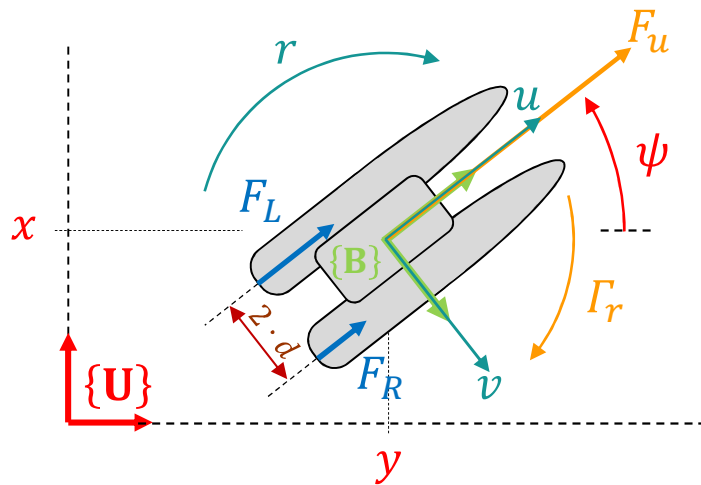
Under-actuation

$$\begin{aligned}
 \begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}_{\dot{\boldsymbol{\eta}}} &= \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \begin{bmatrix} u \\ v \\ r \end{bmatrix}_{\mathbf{v}} \\
 &\rightarrow \dot{\boldsymbol{\eta}} = \mathbf{R} \cdot \mathbf{v}
 \end{aligned}$$

$$\begin{aligned}
 \begin{bmatrix} \dot{u} \\ \dot{v} \\ \dot{r} \end{bmatrix}_{\dot{\mathbf{v}}} &= \begin{cases} \frac{1}{X_{\dot{u}}} \cdot (F_u - X_u \cdot u \cdot |u|) \\ \frac{1}{Y_{\dot{v}}} \cdot (\cancel{F_v} - Y_v \cdot v \cdot |v| - Y_{ur} \cdot u \cdot r) \\ \frac{1}{N_{\dot{u}}} \cdot (F_r - N_r \cdot r \cdot |r|) \end{cases} \\
 &\rightarrow \dot{\mathbf{v}} = \mathbf{f}_D(\boldsymbol{\Theta}, \mathbf{v}, \boldsymbol{\eta}, \mathbf{F}_B)
 \end{aligned}$$

Actuation Model

- 2D, Cartesian, Propulsive actuation, ASV



$$\begin{bmatrix} F_u \\ \Gamma_r \end{bmatrix} = \underbrace{\begin{bmatrix} 1 & 1 \\ d & -d \end{bmatrix}}_{\mathbf{A}} \cdot \underbrace{\begin{bmatrix} F_L \\ F_R \end{bmatrix}}_{\mathbf{F}_m}^T$$

$$\rightarrow \mathbf{F}_B = \mathbf{A} \cdot \mathbf{F}_m$$

$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\boldsymbol{\eta}}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

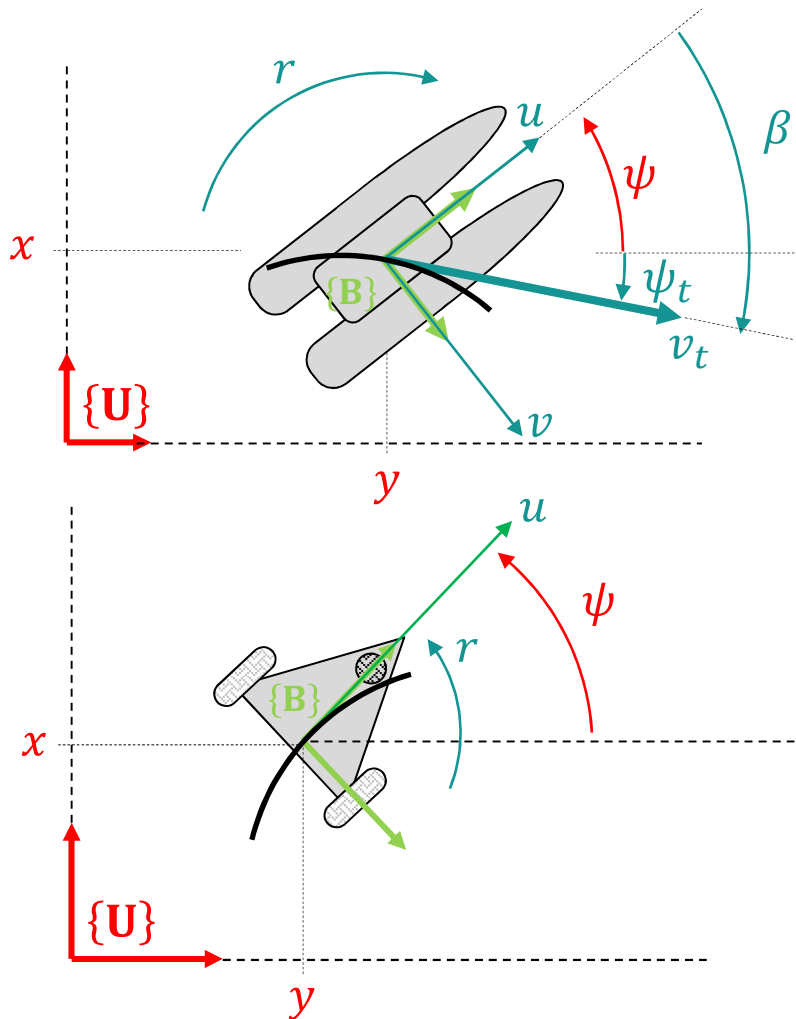
$$\rightarrow \dot{\boldsymbol{\eta}} = \mathbf{R} \cdot \mathbf{v}$$

$$\underbrace{\begin{bmatrix} \dot{u} \\ \dot{v} \\ \dot{r} \end{bmatrix}}_{\dot{\mathbf{v}}} = \begin{cases} \frac{1}{X_{\dot{u}}} \cdot (F_u - X_u \cdot u \cdot |u|) \\ \frac{1}{Y_{\dot{v}}} \cdot (-Y_v \cdot v \cdot |v| - Y_{ur} \cdot u \cdot r) \\ \frac{1}{N_{\dot{r}}} \cdot (\Gamma_r - N_r \cdot r \cdot |r|) \end{cases}$$

$$\rightarrow \dot{\mathbf{v}} = \mathbf{f}_D(\boldsymbol{\Theta}, \mathbf{v}, \boldsymbol{\eta}, \mathbf{F}_B)$$

Back to kinematics

- 2D, Cartesian, ASV vs Unicycle



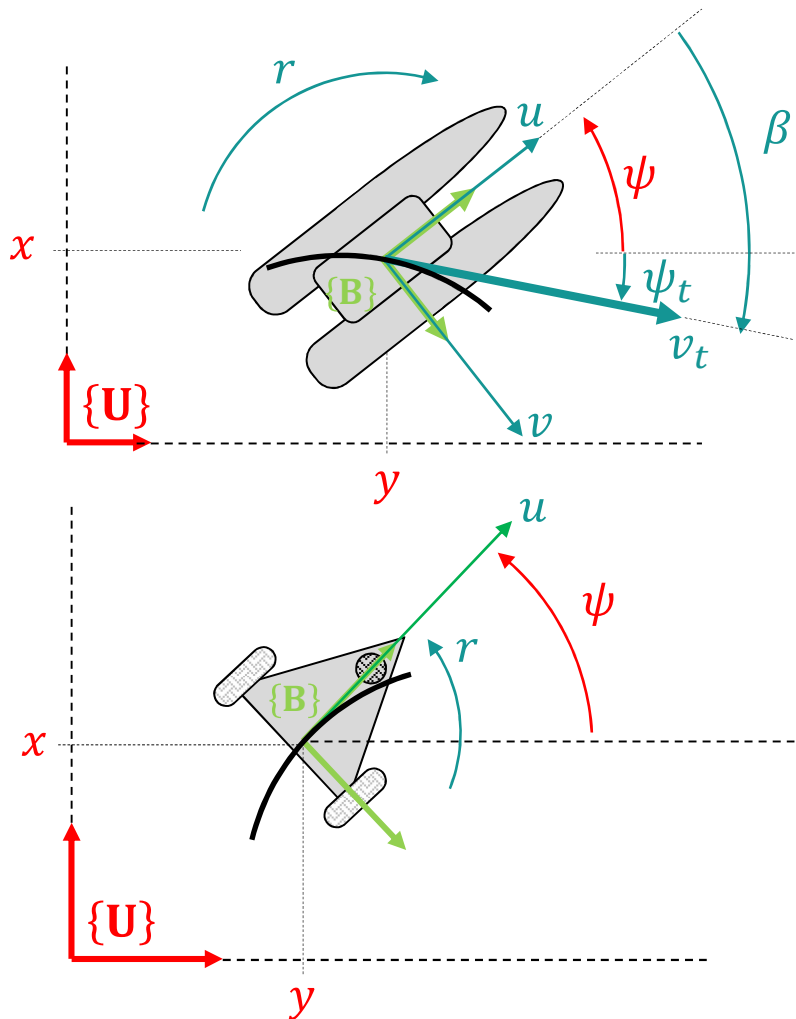
$$\begin{aligned} \underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} &= \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}} \\ &= \begin{bmatrix} \cos \psi_t & 0 \\ \sin \psi_t & 0 \\ 0 & 1 \end{bmatrix} \cdot \begin{bmatrix} v_t \\ \psi_t \end{bmatrix} \end{aligned}$$

The total velocity of a moving object is necessarily tangent to its own trajectory

$$\begin{aligned} \underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} &= \underbrace{\begin{bmatrix} \cos \psi & 0 \\ \sin \psi & 0 \\ 0 & 1 \end{bmatrix}}_{\mathbf{J}} \cdot \underbrace{\begin{bmatrix} u \\ r \end{bmatrix}}_{\mathbf{v}} \end{aligned}$$

Back to kinematics

- 2D, Cartesian, ASV vs Unicycle



$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \begin{bmatrix} \cos \psi_t & 0 \\ \sin \psi_t & 0 \\ 0 & 1 \end{bmatrix} \cdot \begin{bmatrix} v_t \\ \dot{\psi}_t \end{bmatrix}$$

$$\dot{\psi}_t = r + \dot{\beta}, \text{ where } \beta = \text{atan} \frac{v}{u}$$

$$\begin{cases} F_u = m_u \cdot \dot{u} + d_u \\ 0 = m_v \cdot \dot{v} + m_{ur} \cdot u \cdot r + d_v \\ \Gamma_r = m_r \cdot \dot{r} + d_r \end{cases}$$

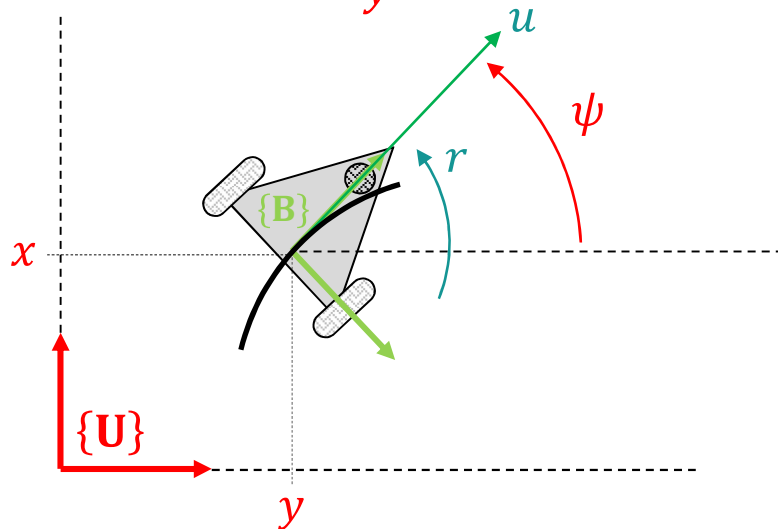
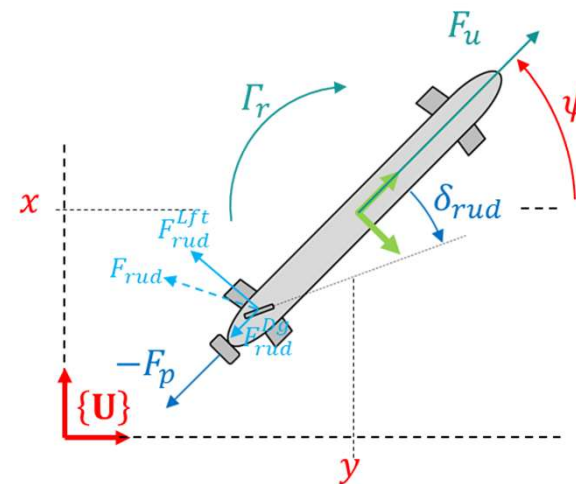
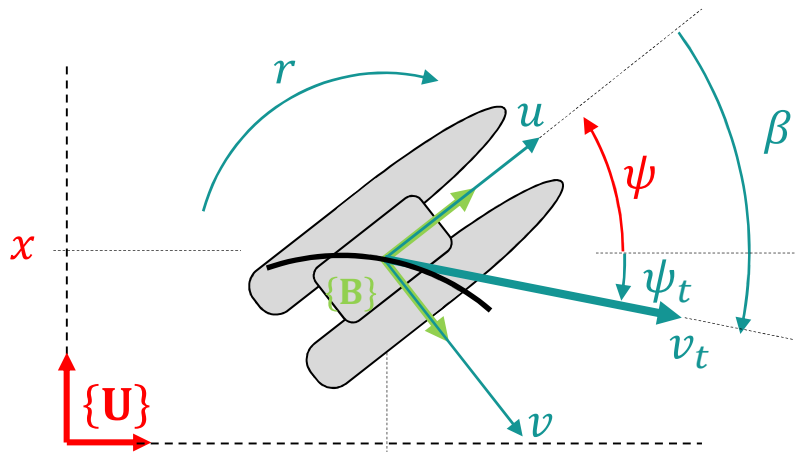
$$\rightarrow r = \frac{\dot{\psi}_t + f(\beta, \mathbf{v}, \dot{u})}{1 - \cos^2 \beta \cdot \frac{m_{ur}}{m_v}}$$

$$\text{well posed if } \frac{m_{ur}}{m_v} = \frac{m - Y_r}{m - Y_{\dot{v}}} < 1$$

Covered by *stern dominance* assumption
(open-loop local stability)

Back to kinematics

- 2D, Cartesian, ASV, AUV vs Unicycle

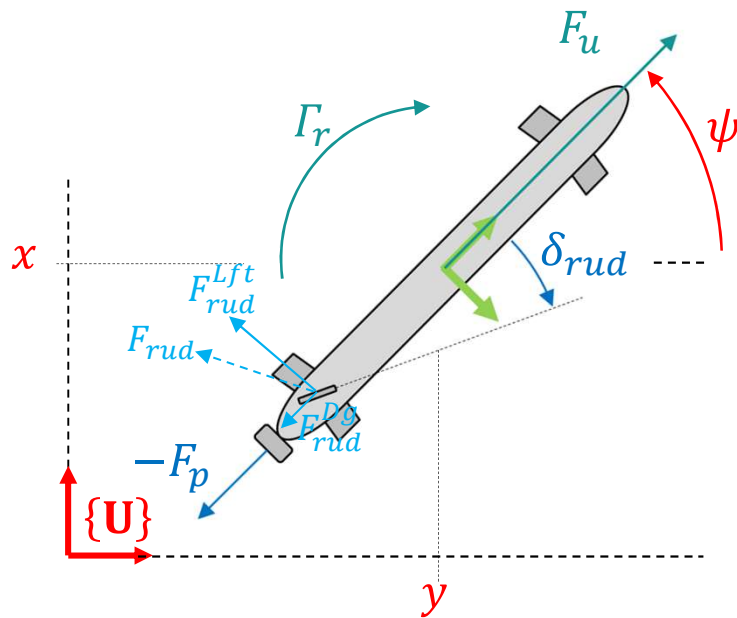


$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\eta}} = \begin{bmatrix} \cos \psi_t & 0 \\ \sin \psi_t & 0 \\ 0 & 1 \end{bmatrix} \cdot \begin{bmatrix} v_t \\ r + \dot{\beta} \end{bmatrix}$$

Covered by *stern dominance* assumption
(open-loop local stability)

Actuation Model

- 2D, Cartesian, Propulsive actuation, AUV



$$\underbrace{\begin{bmatrix} F_u \\ F_v \\ \Gamma_r \end{bmatrix}}_{\mathbf{F}_B} = \underbrace{\begin{bmatrix} 1 & 1 & 0 \\ 0 & 0 & 1 \\ 0 & 0 & d \end{bmatrix}}_{\mathbf{A}} \cdot \underbrace{\begin{bmatrix} F_p \\ F_{rud}^{Dg} \\ F_{rud}^{Lft} \end{bmatrix}}_{\mathbf{F}_m} = \mathbf{f}_{rud}(v_t, \delta_{rud})^T$$

$$\rightarrow \mathbf{F}_B = \mathbf{A} \cdot \mathbf{F}_m$$

$$\underbrace{\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{\psi} \end{bmatrix}}_{\dot{\boldsymbol{\eta}}} = \underbrace{\begin{bmatrix} \cos \psi & -\sin \psi & 0 \\ \sin \psi & \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{\mathbf{R}} \cdot \underbrace{\begin{bmatrix} u \\ v \\ r \end{bmatrix}}_{\mathbf{v}}$$

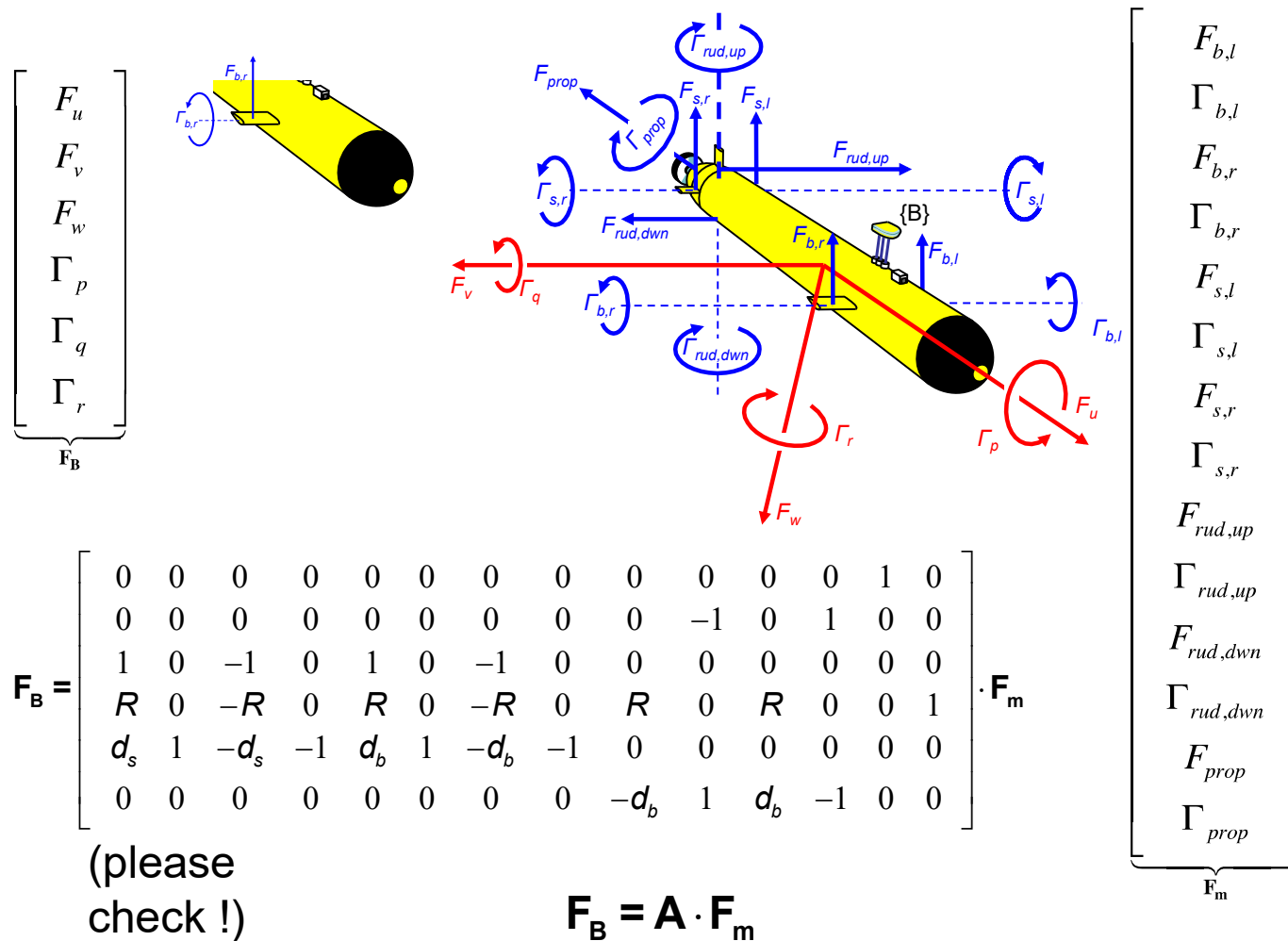
$$\rightarrow \dot{\boldsymbol{\eta}} = \mathbf{R} \cdot \mathbf{v}$$

$$\underbrace{\begin{bmatrix} \dot{u} \\ \dot{v} \\ \dot{r} \end{bmatrix}}_{\dot{\mathbf{v}}} = \begin{cases} \frac{1}{X_{\dot{u}}} \cdot (F_u - X_u \cdot u \cdot |u|) \\ \frac{1}{Y_{\dot{v}}} \cdot (F_v - Y_v \cdot v \cdot |v| - Y_{ur} \cdot u \cdot r) \\ \frac{1}{N_{\dot{r}}} \cdot (\Gamma_r - N_r \cdot r \cdot |r|) \end{cases}$$

$$\rightarrow \dot{\mathbf{v}} = \mathbf{f}_D(\boldsymbol{\Theta}, \mathbf{v}, \boldsymbol{\eta}, \mathbf{F}_B)$$

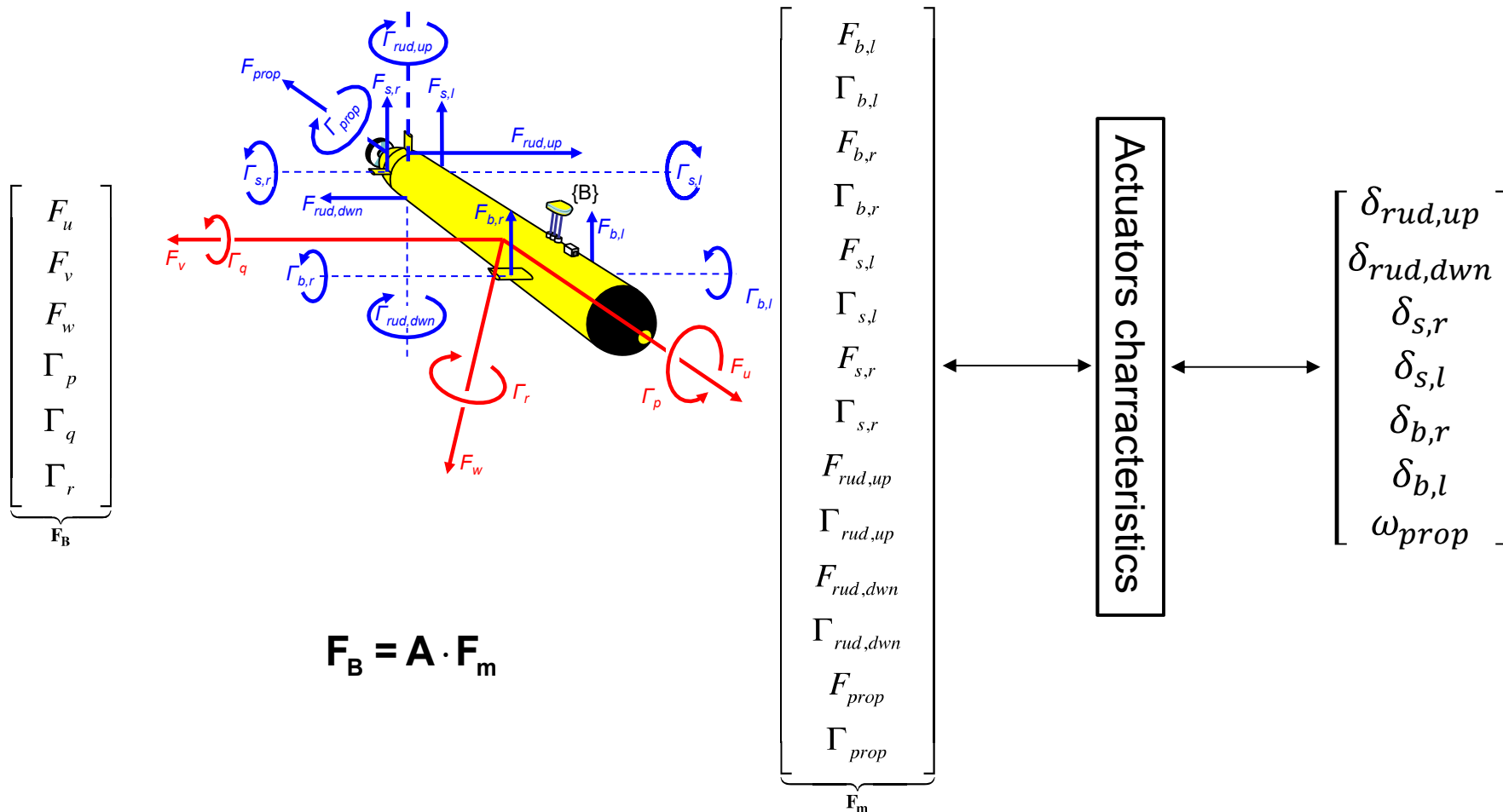
Actuation Model

- 3D, Cartesian, Propulsive actuation, AUV



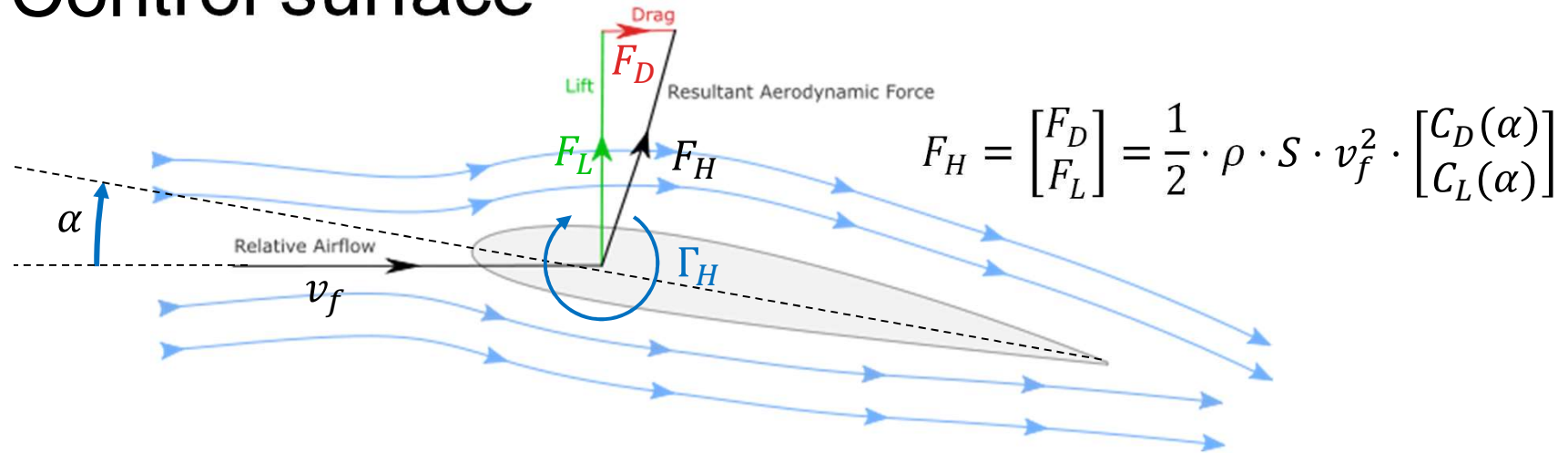
Actuation Model

- 3D, Cartesian, Propulsive actuation, AUV

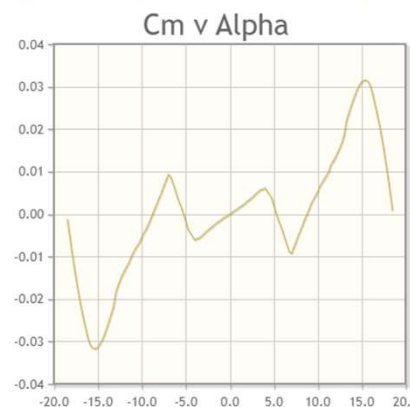
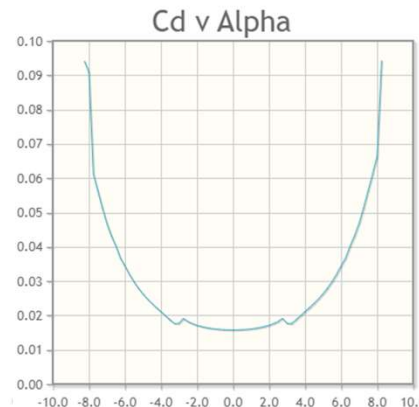
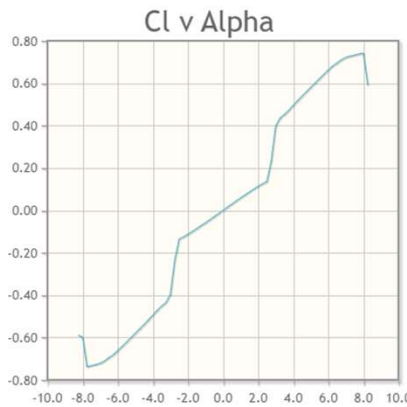
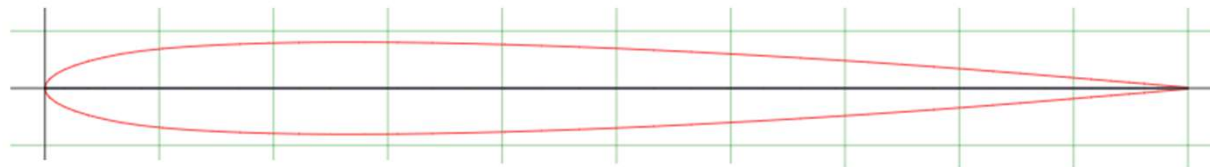


Actuator characteristics

- Control surface



$$F_H = \begin{bmatrix} F_D \\ F_L \end{bmatrix} = \frac{1}{2} \cdot \rho \cdot S \cdot v_f^2 \cdot \begin{bmatrix} C_D(\alpha) \\ C_L(\alpha) \end{bmatrix}$$

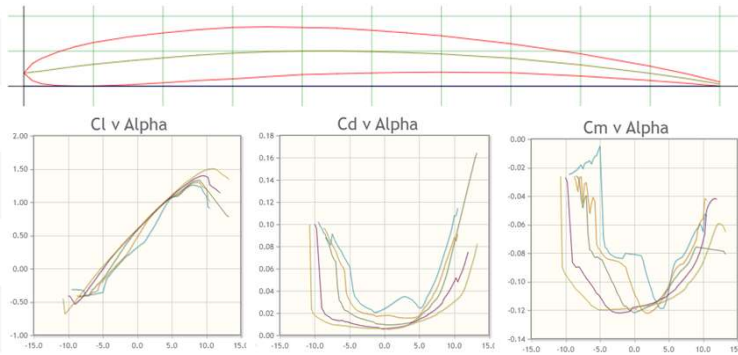


$$\begin{bmatrix} F_D \\ F_L \\ \Gamma_H \end{bmatrix} = \mathbf{f}_m(\delta(c_m))$$

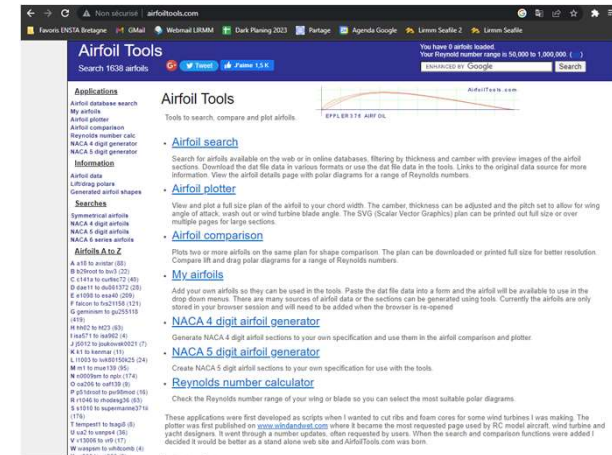
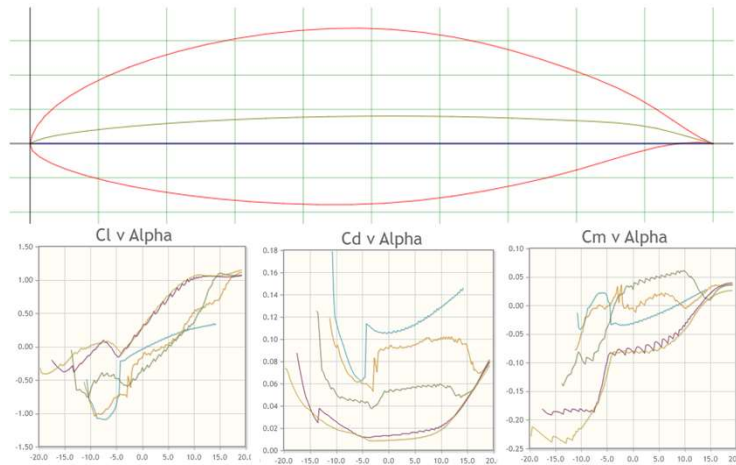
Actuator characteristics

- Control surface

Plot	Airfoil	Reynolds #
<input checked="" type="checkbox"/>	a18-il	50,000
<input type="checkbox"/>	a18-il	50,000
<input checked="" type="checkbox"/>	a18-il	100,000
<input type="checkbox"/>	a18-il	100,000
<input checked="" type="checkbox"/>	a18-il	200,000
<input type="checkbox"/>	a18-il	200,000
<input checked="" type="checkbox"/>	a18-il	500,000
<input type="checkbox"/>	a18-il	500,000
<input checked="" type="checkbox"/>	a18-il	1,000,000
<input type="checkbox"/>	a18-il	1,000,000

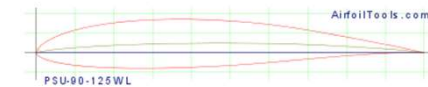


Plot	Airfoil	Reynolds #
<input checked="" type="checkbox"/>	th25816-il	50,000
<input type="checkbox"/>	th25816-il	50,000
<input checked="" type="checkbox"/>	th25816-il	100,000
<input type="checkbox"/>	th25816-il	100,000
<input checked="" type="checkbox"/>	th25816-il	200,000
<input type="checkbox"/>	th25816-il	200,000
<input checked="" type="checkbox"/>	th25816-il	500,000
<input type="checkbox"/>	th25816-il	500,000
<input checked="" type="checkbox"/>	th25816-il	1,000,000
<input type="checkbox"/>	th25816-il	1,000,000



<http://airfoiltools.com/airfoil/naca4digit>

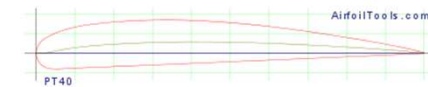
(psu-90-125wl-il) PSU-90-125WL



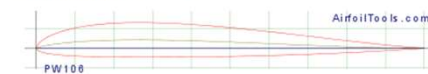
(psu94097-il) PSU 94-097 (AIAA 2001-2478)



(pt40-il) PT40



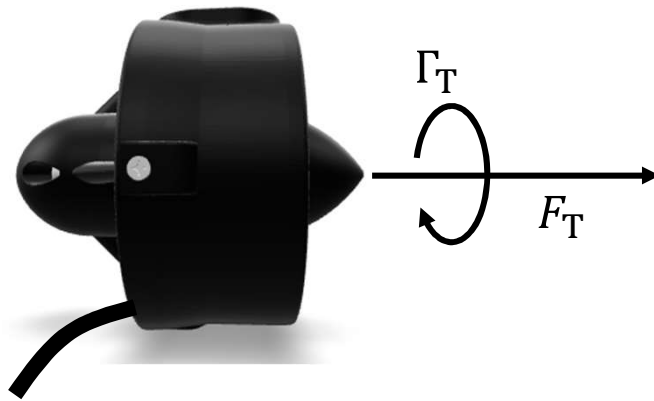
(pw106-pw) PW106



$$\mathbf{F}_m = [F_D, F_L, \Gamma_M]^T = \mathbf{f}_m(C_L(\alpha), C_D(\alpha), C_M(\alpha), \nu, \delta(c_m), S, \rho)$$

Actuator characteristics

- Thruster (T200, BlueRobotics)



$c_m(\text{PWM})$

$$\begin{bmatrix} F_T \\ \Gamma_T \end{bmatrix} = \mathbf{f}_m(c_m)$$

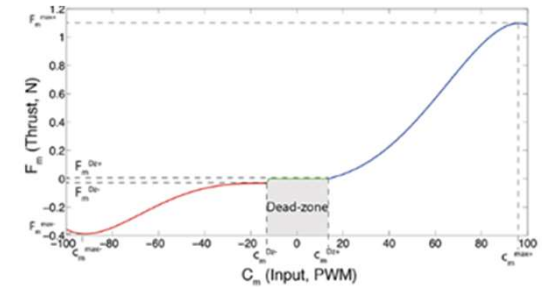
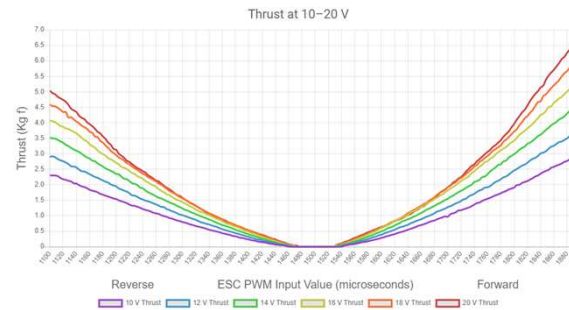
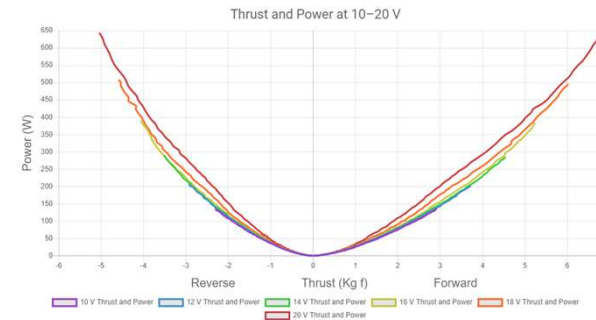
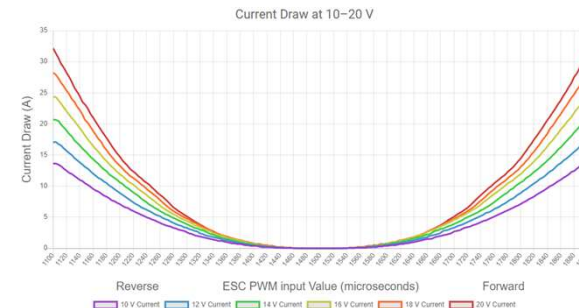


Fig. 4. Motor characteristic identification.



Actuator characteristics

- Thruster (T200, BlueRobotics)

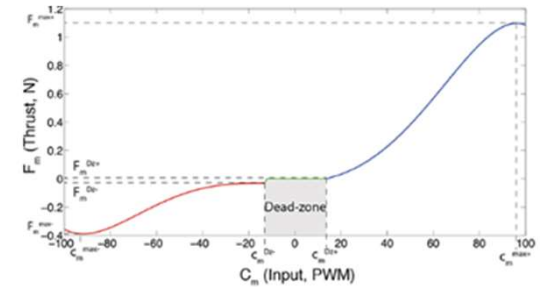
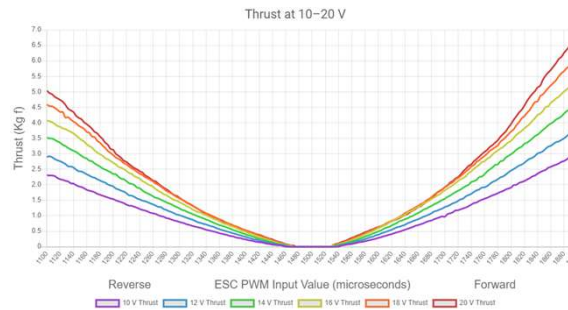
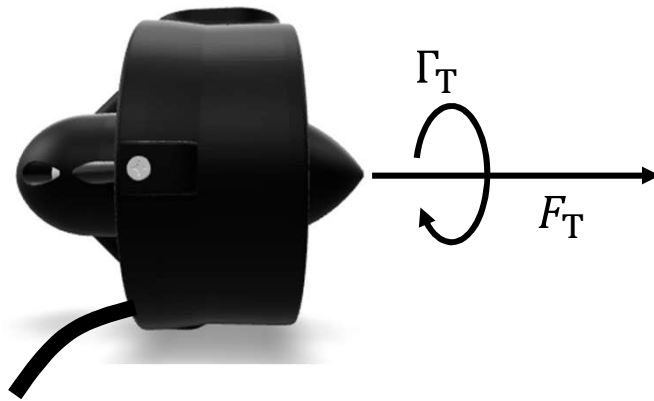
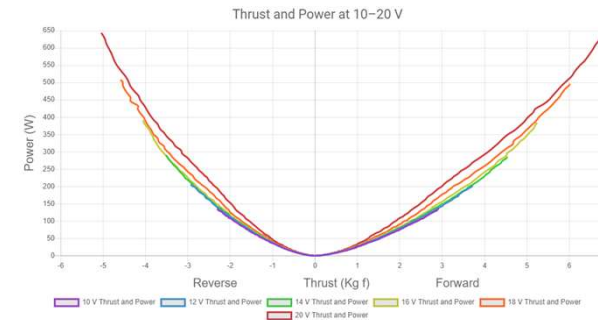
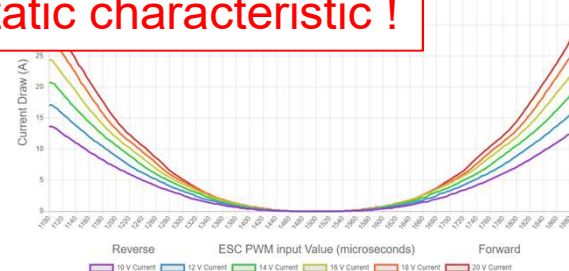


Fig. 4. Motor characteristic identification.

! Static characteristic !

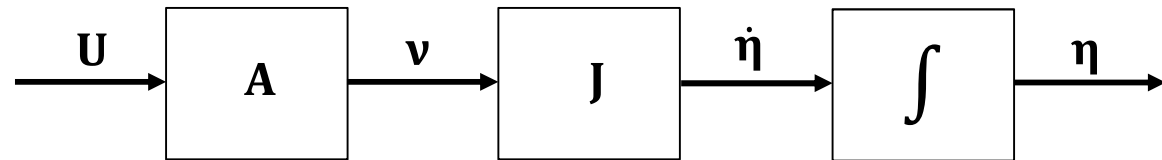


$c_m(\text{PWM})$

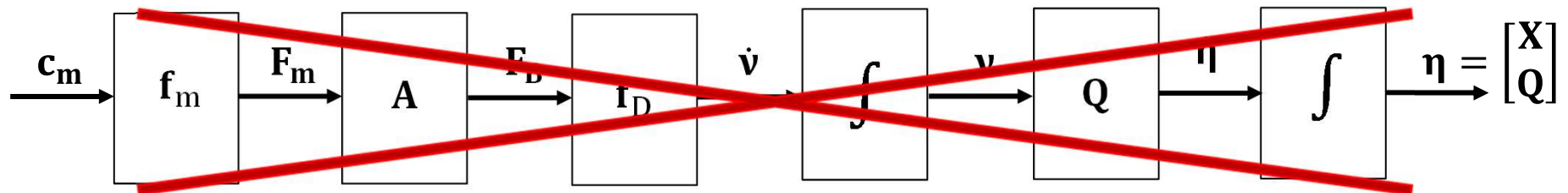
$$\begin{bmatrix} F_T \\ \Gamma_T \end{bmatrix} = \mathbf{f}_m(c_m)$$

Simulator

- Kinematics

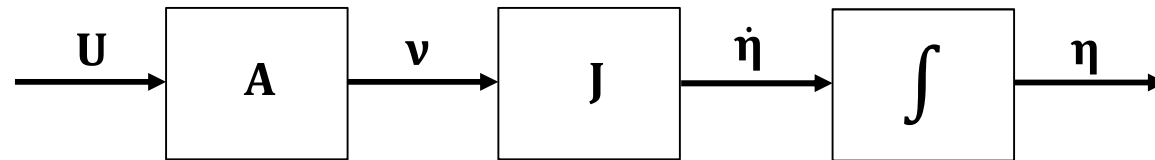


- Dynamics



Simulation

- Kinematics



- Dynamics

